

# Supercritical thermodynamic property evaluation via adaptive mesh tabulation

by

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## Abstract

Obtaining accurate computational simulations of fluid flows with a complex thermodynamic behaviour is difficult. This is partly due to the non-linear variation of thermophysical properties in the vicinity of critical point. Supercritical fluid flows are observed in a wide range of applications such as caffeine extraction, supercritical diesel fuel injection, nuclear reactors, and liquid rocket engines. In the near-critical regime, thermodynamic properties are accurately described by a multi-parametric equation of state, but most often using such complex state equations involve expensive computations, thereby consuming a significant portion of the available computational resources. To mitigate this issue, tabulation of state equations stands as an alternative yet they are inefficient due to the strong non-linear property variation in the proximity of critical point. In this thesis a novel tabulation method based on adaptive mesh refinement (AMR) approach is presented, it enabled accurate thermodynamic and physical property evaluation with minimal computational effort. Detailed analyses in the form of error validation, computational cost comparison were performed with reference to the commonly used cubic equations of state. In order to demonstrate the grid scaling effects on total computational cost of a real-fluid CFD simulation, a one dimensional harmonic acoustic wave case was chosen. We show a significant computational cost reduction with the adaptive tabulation approach relative to the cubic state equations (with an underlying iterative root-finding method). It also offered an accurate emulation of the backend equation of state with significantly less computation cost.

The developed adaptive tabular equation of state is also integrated into a Computational Fluid Dynamics (CFD) code which has numerical techniques enabling computation of flows with large density gradients. To validate this CFD solver and to observe accurate thermodynamics effects in gasdynamics simulations, the Sod-shock tube and Shu-Osher shock tube problems were solved computationally for both perfect, real fluid thermodynamics. Also the sod-shock tube problem was simulated with three different thermodynamic initial conditions (supercritical, nearcritical, subcritical regions) the nearcritical case has shown a relatively smaller change in temperature, pressure at the shock contact while having a large change in density compared to two other scenarios. It represented the underlying supercritical thermodynamic behaviour that the density increases drastically with a relatively small change in temperature.

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Last but most importantly, I would like to give my greatest appreciation to my parents, friends, cousins for their unconditional support.

“Today is very difficult. Tomorrow is even more difficult. But the day after tomorrow is very beautiful. Most people die tomorrow evening. If you don't work hard, you don't see the sunshine.. *Never give up !* ” - Jack Ma.

## **Dedication**

To my beloved parents

అమ్మ , నాన్న

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# List of Abbreviations

**CFD** Computational Fluid Dynamics

**RK** Redlich-Kwong Equation of State

**BWR** Benedict-Webb-Rubin Equation of state

**mBWR** Modified Benedict-Webb-Rubin Equation of state

**PR** Peng-Robinson Equation of state

**SRK** Soave-Redlich Kwong Equation of state

**GPU** Graphical Processing Units

**AMR** Adaptive Mesh Refinement

**HLLC** Harten-Lax-van Leer-Contact Riemann solver

**PDE** Partial Differential Equation

**ODE** Ordinary Differential Equation

**EOS** Equation of state

**CoolProp** An open source thermodynamics and transport property database

**NIST** National Institute of Standard and Technology

**REFPROP** NIST Reference Fluid Thermodynamic and Transport Properties Database

**WENO** Weighted Essentially Non-Oscillatory

# Chapter 1

## Introduction

### 1.1 Overview

Matter exists in three different phases: solid, liquid and gas [53]. The boundaries between these phases are best illustrated with the help of a phase diagram, shown in figure 1.1. The equilibrium phase diagram for a single-species allows us to determine the matter's current phase based on, at most, two intensive thermodynamic properties such as: pressure, and temperature [35]. The dividing lines represent the phase boundaries and thereby indicate the physical processes of boiling, melting and sublimation. There are two special points in a phase diagram, namely: triple point and *critical point*. The triple point is a thermodynamic singularity in which all the three phases coexist in equilibrium [66]. Whereas, the critical point is an endpoint in the phase equilibrium curve between liquid and gas [12]. At a temperature and pressure above the critical point, one cannot differentiate the state of matter (at least not using classical metrics); in this state, we are in a supercritical regime [37]. Matter in this supercritical regime has distinct physical properties such as large compressibility and a high diffusivity (similar to gases). This unusual behaviour is exploited in various physical processes such as fluid extraction [48], separation and fuel injection. For example, supercritical diesel injection provides better air-fuel mixing due to the high molecular diffusivity (compared to liquid injection) and enables a better control over the engine operation [5]. Another property of the supercritical regime is low viscosity and non-existent surface tension [37] due to the fact there is no clear delineation between the liquid, and gas states. Several industrial applications such as Organic Rankine Cycle turbines, supercritical CO<sub>2</sub> power systems, transcritical heat exchangers, liquid propellant rocket engines[] are associated with near-critical, transcritical or supercritical flow regimes.

In order to improve these systems efficiencies and assure stable operation, accurate study of the flow physics is essential. Due to the cost and complexities involved in performing experiments with supercritical fluids, researchers and engineers have adopted Computational Fluid Dynamics (Computational Fluid Dynamics (CFD)) simulations as an analysis tool for design in many of these industrial applications.

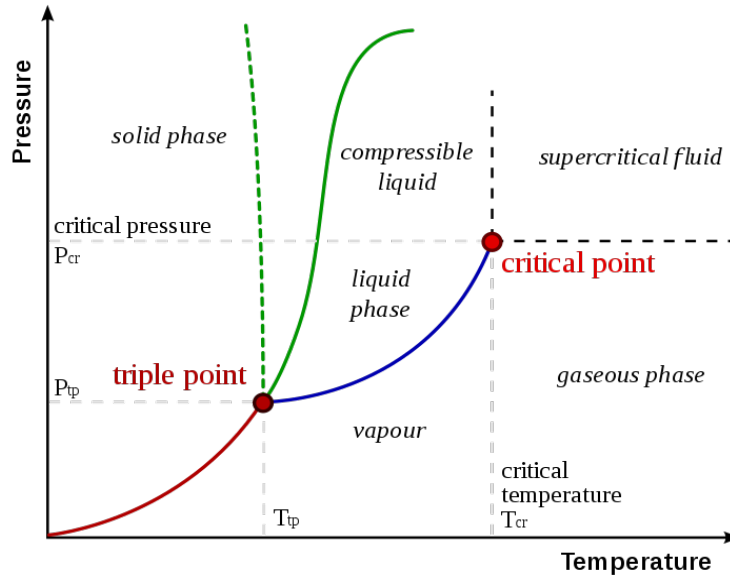


Figure 1.1: Phase diagram: green line, blue line, red line delineate the solid-liquid, liquid-vapour, solid-vapour transition lines, respectively. Source [1]

It is highly desirable to have an accurate, yet robust, CFD simulation for supercritical thermodynamic applications. But CFD simulations involving such pseudo-phase transitions are inherently unstable [44] not only due to the large density gradient near the critical point but also, due to the non-linearity of the thermophysical properties resulting in spurious oscillations[47] in the numerical flux computations. Figure 1.2 shows the sharp change in specific heat  $C_p$  at supercritical pressures for oxygen. Another aspect that inhibits the fidelity of such simulations is the accuracy of the equation of state (EOS) that is being used in the CFD solver. The ideal gas law is the most commonly used EOS but the inherent assumptions of this state equation break down in the supercritical thermodynamic regime. In such scenarios, the EOS must account for the intermolecular forces in the gas. For computational efficiency, CFD practitioners usually incorporate a cubic equation of state such as Redlich-Kwong Equation of State (RK) [52], Peng-Robinson PR



[49], Soave-Redlich Kwong [SRK](#) [58], all of which are derived from the Van der Waals equation, to estimate the thermodynamic properties of the fluid. But these cubic equations have nearly 10-15 % relative errors [62] as we approach critical point as shown in figure 1.3 when compared to a higher-order state equation such as the modified Benedict Webb Rubin Modified Benedict-Webb-Rubin Equation of state ([mBWR](#)) [59] equation of state. The [mBWR](#) has 32 parameters, 20 of which describe the saturation curve and its critical point. It matches suitably with experimentally obtained thermodynamic property data and is generally used to interpolate experimental data in thermodynamic databases such as [CoolProp](#), [REFPROP](#).

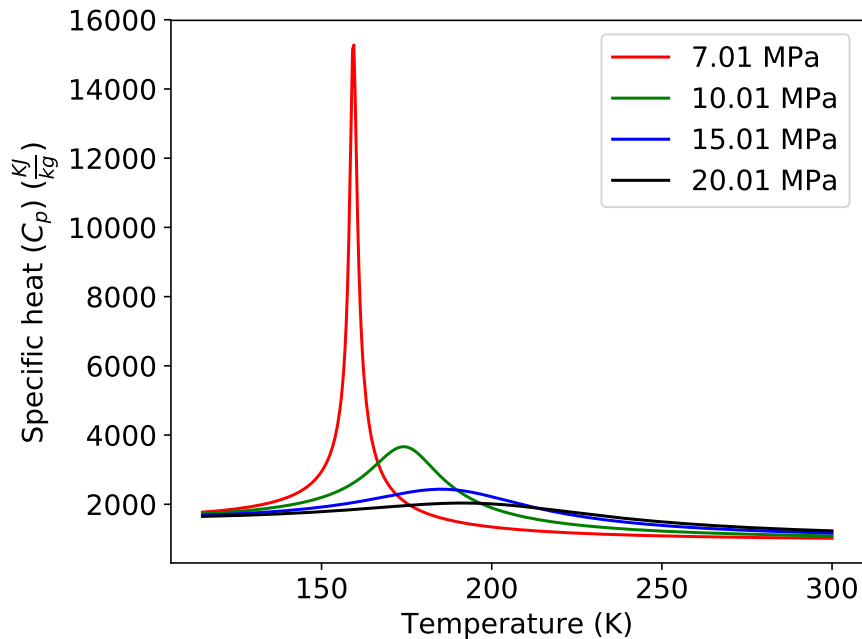
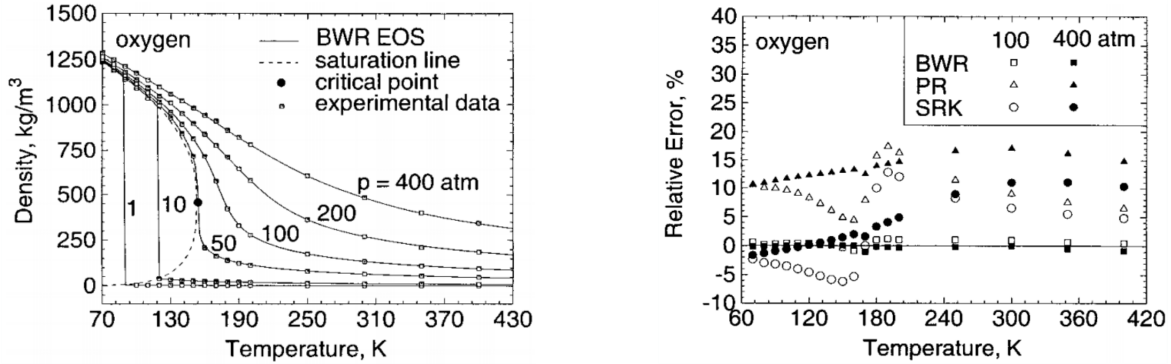


Figure 1.2: Isobaric specific heat ( $C_p$ ) of oxygen.

In a typical [CFD](#) solution process, the compressible Navier-Stokes equations are closed using an equation of state [EOS](#). In other words, given the conserved quantities such as  $[\rho, \rho \vec{u}, \rho E]$ , the [EOS](#) allows us to obtain the primitive variables pressure ( $P$ ) and temperature ( $T$ ), the gradients of which are needed to compute the Navier-Stokes equations. The key numerical difficulty in using a highly accurate non-linear [EOS](#) lies in computing the primitive thermodynamic variables, as the state equations generally take the form of  $P = f(\rho, T)$  and are not a function of  $f(\rho, e)$ . In the supercritical regime, the internal

energy of a fluid depends on two independent thermodynamic variables. In order to find approximate thermodynamic solution, iterative numerical methods such as Newton-Raphson solvers are used. One way to mitigate the cost of an iterative root-finding method is to build a pre-tabulated look-up table [50, 73] where the CFD solver looks-up the thermodynamic properties during run-time. However, the complexity lies in building such table in the transcritical critical regime, as we can see from the figure 1.2 the thermodynamic property gradient is highly non-linear near the critical point. Hence, any such look-up table would have to consider this abrupt change in gradient (either Pressure-Temperature or Density - Internal Energy).



(a) Oxygen density computed using Benedict-Webb-Rubin (BWR) Equation of State compared with experimental data

(b) Relative errors of density as determined by three different Equations of state with reference to Experimental data

Figure 1.3: Comparison of various polynomial equations of state with experimental data from Sychev et al. [62]

## 1.2 Motivation

The scale and complexity of computational fluid dynamics simulations are ever increasing, compared to 800,000 mesh points in 1970s to more recent direct numerical simulation of isotropic turbulence [32] with  $4096^3 = 68$  billion grid points there is a huge advancement in terms of hardware and numerical methods that led to accelerated numerical convergence rates and stable computational algorithms [56]. The need for an accurate real fluid thermodynamic property evaluation in transcritical, supercritical CFD simulations had been suggested by several researchers [13, 30, 63, 44, 36]. A strong influence of real gas effects for high pressure supersonic methane jets, hydrogen jets was demonstrated by Hempert et

al. [30] and Bonelli et al. [13], as shown in figure 1.4 there was a difference in shockfront width, a very small shockfront was recognizable in the case of real fluid while the absence of real fluid thermodynamics had wider shockfronts. However, the current simulations involving accurate real fluid thermodynamics are very expensive not only due to the number of chemical species involved, but also due to the inherent complexity in terms of their properties evaluation based iterative numerical methods such as computing compressibility factor (Z) using Newton-Raphson solver. Some of the widely used multi-parametric cubic equation of states, such as SRK and PR, were invented for the sole purpose of evaluating the hydrocarbon mixtures at low and high pressures. Even after 40 years of advancements in both hardware and computational methods, relying on the same old method of iterative root-finding to perform high fidelity simulations is a matter of concern. Hence, it is very important to research new approaches/algorithms towards developing computationally efficient yet, accurate method of thermodynamic properties evaluation, which can also be incorporated into the modern CFD codes with minimal effort.

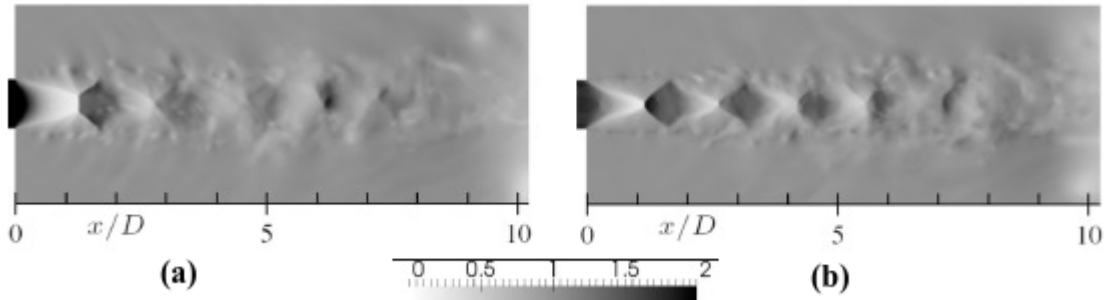


Figure 1.4: Pressure ratio  $\frac{P_{in}}{P_{out}}$  for a supersonic jet, LES simulation of high pressure supersonic jet [30] with (a) Ideal gas thermodynamics (b) Real fluid thermodynamics

Thermodynamic and thermophysical property evaluations often consume the majority of computational resources in any CFD simulation involving real fluid thermodynamics. For example in a LES [25] of a supercritical fuel-jet in a cross flow, real fluid thermodynamic properties are considered and evaluated using PR EOS and computations are accelerated with a Graphical Processing Units (GPU). The profiling data is illustrated in the figure 1.5, 1.6. They show the fraction of time spent in various modules, sub-routines to the fraction of total computation time. From figure 1.5 we can see that the thermodynamic-transport closures comprise 51.2% of total computation time yet they occupy 24.1 % of total floating operations. Thereby making single floating point operation of a thermodynamics-transport module significantly higher than other routines. Also, if

we inspect thermodynamic-transport module more specifically (figure 1.6), the Z-factor sub-routine (compressibility factor calculation) has a big influence on the total floating point operations. Being able to lookup such computationally intensive operations during the run-time would significantly enhance the solver’s capabilities.

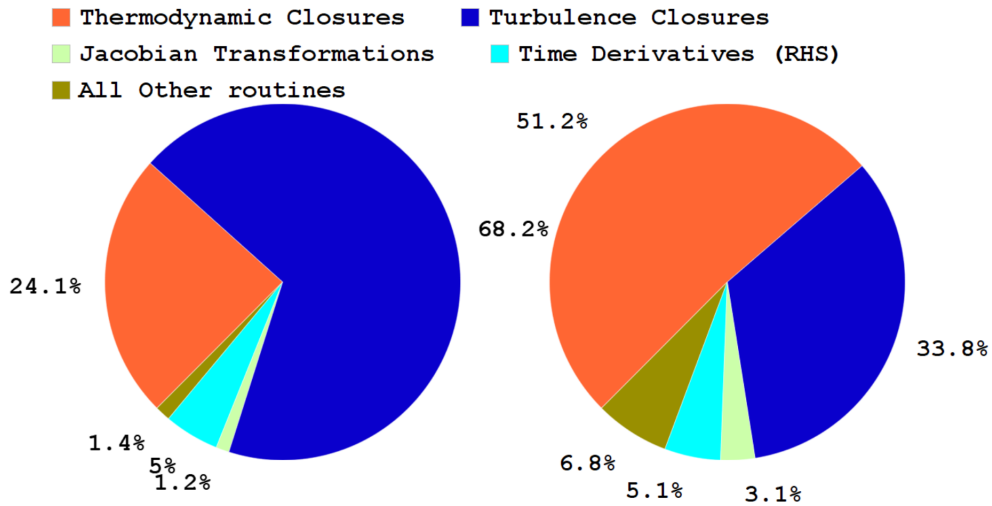


Figure 1.5: (a) Floating point operations as a percent of total flops for all routines (left) (b) Time for computation of different routines/modules as a percent of total computational time (right). Data obtained from an LES simulation of supercritical fuel jet [25].

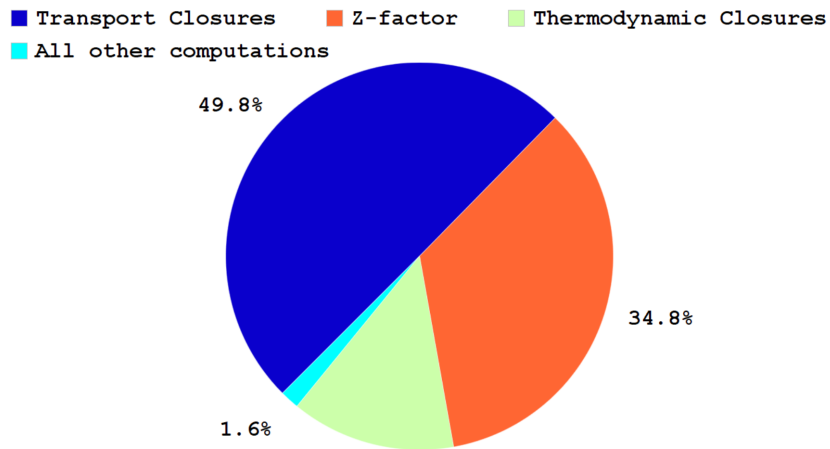


Figure 1.6: Floating point operations for different thermodynamic and transport routines[25].

## 1.3 Objectives

The objective of this work is to formulate a computationally efficient and accurate tabulated thermodynamic equation of state and thermophysical properties for [CFD](#) solvers and investigate its performance with reference to the current existing equation of states such as cubic, multiparametric in terms of speed and accuracy. Further, we implemented the developed [EOS](#) into a [CFD](#) code to simulate real fluid thermodynamic effects in gas dynamic test cases.

### 1.3.1 Contributions

The main contributions of this thesis are:

- A tabulated equation of state using block-structured adaptive mesh refinement ([AMR](#)) technique. Validation against the conventional methods of property evaluation such as cubic, multi-parametric [EOS](#) for speed and accuracy. Demonstration of computational bottlenecks in using a cubic [EOS](#) for real-fluid simulations, with the help of a one-dimensional harmonic acoustic wave test case.
- Extended an existing [CFD](#) code to enable real-fluid simulations, added features include: tabular [EOS](#) integration, adaptation of Harten-Lax-van Leer-Contact Riemann solver ([HLLC](#)) to real fluid inviscid flux evaluation, fourth-order accurate Runge-Kutta time advancement scheme with frozen specific heat ratio to prevent spurious oscillations due to large density gradients.
- The significance of using an accurate thermodynamic properties evaluation in [CFD](#) simulations is studied with the help of two gasdynamics test cases (Sod, Shu-Osher shock tube). Three different scenarios i.e., Supercritical, nearcritical, subcritical initial conditions were employed to observe non-classical gas dynamic behaviour.

## 1.4 Outline

This thesis is organized into six chapters, starting with chapter [2](#) on essential mathematical background with regard to underlying governing equations and thermodynamics. Chapter [3](#) details the tabulated thermodynamic [EOS](#) formulation using block-structured adaptive mesh refinement. Chapter [4](#) describes the [CFD](#) solver that is used to test this

tabulated [EOS](#) and explains the intricacies of integrating a real fluid thermodynamic [EOS](#) into a Riemann solver. Using this combination of tabulated [EOS](#) and [CFD](#) solver, chapter [5](#) investigates gas dynamic test cases in three different thermodynamic regimes namely: supercritical, near-critical and subcritical. Conclusions and future work are presented in chapter [6](#).

Chapter [2](#) provides a detailed overview of compressible flow governing equations, their properties such as characteristic form, Eigen-values. The concept of equation of state is explained and various types of existing [EOS](#) such as cubic, multiparametric, thermodynamic databases are discussed in detail. The calculation of thermodynamic properties such as enthalpy, entropy using departure function is presented.

Chapter [3](#) presents a detailed framework on tabulated real fluid thermodynamics. Starting with the simplest form of tabulation i.e., homogeneous or logarithmic tabulation, various other forms of tabulation such as Block structured adaptive mesh refinement [AMR](#) and Cubic Bezier surface based adaptive mesh refinement [AMR](#). The table generation process in each type of approach is discussed in detail. One of the important aspect in the tabulation of thermodynamics is its storage and retrieval (look-up) process, which will be discussed in this chapter. Also, tabulated [EOS](#) thermodynamic consistency evaluation is discussed. Further error estimates, speed-up in computational time when such tabular [EOS](#) is implemented in a [CFD](#) solver is discussed with the help of an 1D acoustic wave simulation.

Chapter [4](#) details the numerical schemes, specialties of the [CFD](#) code (TwoDThermoCode) that is used for this study, starting with the computational domain discretization of governing equations using finite volume method in an explicit form, we will discuss on obtaining solution to the Riemann problem, refraction & shocks. For this code, second order accurate reconstruction of cell interface states is achieved through piecewise parabolic method. The main objective of this chapter is to discuss the method of integrating real fluid thermodynamics into the [HLLC](#) Riemann solver to compute inviscid fluxes. Various numerical methods to overcome spurious oscillations are explained, and a fourth order accurate Runge-Kutta time advancement implementation is described.

Chapter [5](#) couples both the tabulated thermodynamic equation of state framework and the [CFD](#) solver together to study the real gas effects on compressible flow, two test cases are used for this study: the Sod shock tube and the Shu-Osher shock tube. A validation result is obtained by simulating a near critical test case taken from Terashima [\[64\]](#). Three different thermodynamic initial conditions (subcritical, supercritical and near critical region) are simulated and the real gas effects on shocks/expansion fans are observed.

# Chapter 2

## Mathematical Background

### 2.1 Compressible flow governing equations

The governing equations of unsteady, inviscid, compressible flow describing the macroscopic motion of fluid is the following non-linear hyperbolic partial differential equation set, also known as the Euler equations

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{\mathbf{u}}) = 0 \quad (2.1)$$

$$\frac{\partial \rho \vec{\mathbf{u}}}{\partial t} + \nabla \cdot (\rho \mathbf{u} \mathbf{u}) + \nabla \cdot (\vec{p}) = 0 \quad (2.2)$$

$$\frac{\partial \rho E}{\partial t} + \nabla \cdot (\rho E \vec{\mathbf{u}} + \vec{p} \cdot \vec{\mathbf{u}}) = 0 \quad (2.3)$$

Here,  $E$  is the total energy per unit mass which includes the specific internal energy,  $e$ , and kinetic energy of the flow:

$$E = e + \frac{1}{2} |\vec{\mathbf{u}}|^2 \quad (2.4)$$

Similarly for enthalpy, we have:

$$H = h + \frac{1}{2} |\vec{\mathbf{u}}|^2 \quad (2.5)$$

When viscous and heat transfer effects are included, the governing equation set takes on some features of a parabolic partial differential equation; formally, this equation set is referred to as the Navier-Stokes equations. In  $d$  space dimensions <sup>1</sup>, we have up to

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<sup>1</sup>In this thesis, we consider  $d = 1, 2$

$d + 3$  variables for a single species flow. For example, in 1D we have,  $[\rho, u, p, E]$  and in 2D we have,  $[\rho, u, v, p, E]$  and so on. We have  $d + 2$  equations in the form of continuity (2.1), conservation of momentum(2.2), conservation of energy(2.3) and an equation relating primitive variables such as temperature ( $T$ ), pressure ( $p$ ), density ( $\rho$ ) to internal energy ( $e$ ), often referred as the thermal equation of state serves as a constraint and thereby closes the problem.

Assuming a working fluid as an inviscid and non heat conducting, the relation where pressure ( $p$ ) is a function of specific internal energy ( $e$ ) and density ( $\rho$ ), or mathematically

$$p = p(e, \rho) \quad (2.6)$$

We define the thermodynamic derivatives  $\gamma(e, \rho), \kappa(e, \rho)$  in order to simplify the flux vectors.

$$\gamma(e, \rho) \equiv \frac{\partial p(e, \rho)}{\partial \rho}, \quad \kappa(e, \rho) \equiv \frac{\partial p(e, \rho)}{\partial e}, \quad dp(e, \rho) = \left(\frac{\partial p}{\partial \rho}\right)_e + \left(\frac{\partial p}{\partial e}\right)_\rho \frac{\partial e}{\partial \rho} \quad (2.7)$$

### 2.1.1 Flux formulations in Cartesian form

In two dimensions, the equations 2.1-2.3 can be written in Cartesian coordinates  $(x, y)$  as follows,

$$\frac{\partial \rho}{\partial t} + \frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} = 0 \quad (2.8)$$

$$\frac{\partial(\rho u)}{\partial t} + \frac{\partial(\rho u^2 + p)}{\partial x} + \frac{\partial(\rho uv)}{\partial y} = 0 \quad (2.9)$$

$$\frac{\partial(\rho v)}{\partial t} + \frac{\partial \rho uv}{\partial x} + \frac{\partial(\rho v^2 + p)}{\partial y} = 0 \quad (2.10)$$

$$\frac{\partial \rho E}{\partial t} + \frac{\partial u(\rho E + p)}{\partial x} + \frac{\partial v(\rho E + p)}{\partial y} = 0 \quad (2.11)$$

The above conservative system of equations can be written as follows

$$U_t + (F^x(U))_x + (F^y(U))_y = 0 \quad (2.12)$$



where,

$$U = \underbrace{\begin{pmatrix} \rho \\ \rho u \\ \rho v \\ \rho E \end{pmatrix}}_{\text{Conservative Variables}} \quad F^x(U) = \underbrace{\begin{pmatrix} \rho u \\ \rho u^2 + p \\ \rho v u \\ u \rho E + p u \end{pmatrix}}_{\text{Flux vectors in x-direction}} \quad F^y(U) = \underbrace{\begin{pmatrix} \rho v \\ \rho v^2 + p \\ \rho u v \\ v \rho E + p v \end{pmatrix}}_{\text{Flux vectors in y-direction}} \quad (2.13)$$

This conservative form of the equations is symbolically expressed in the form [68]

$$U_t + \mathbf{J}^x U_x + \mathbf{J}^y U_y = \mathbf{0} \quad (2.14)$$

where,  $\mathbf{J}^x$ ,  $\mathbf{J}^y$  are the Jacobian matrices. Rewriting the flux vectors  $F(U)$  in terms of conservative variables such as internal energy and density, with the help of equation of state, thermodynamic derivatives (2.7), in doing so we represent pressure ( $P$ ) in terms of the internal energy ( $e$ ), density ( $\rho$ ), derivatives of pressure with density ( $\frac{\partial p(e, \rho)}{\partial \rho} \equiv \kappa(e, \rho)$ ) and internal energy ( $\frac{\partial p(e, \rho)}{\partial e} \equiv \gamma(e, \rho)$ ). and thereby, the flux vectors in 2.13 are written as

$$F^x(U) = \begin{pmatrix} \rho u \\ \rho u^2 + p(e, \rho) \\ \rho v u \\ \rho u E + p(e, \rho) u \end{pmatrix} \quad (2.15)$$

$$F^y(U) = \begin{pmatrix} \rho v \\ \rho v^2 + p(e, \rho) \\ \rho u v \\ \rho v E + p(e, \rho) v \end{pmatrix} \quad (2.16)$$

We can obtain a Jacobian matrix by taking the partial derivatives of all individual vectors in a given vector function, here in this case for the hyperbolic system of equations 2.12 with flux vectors  $F^x(U), F^y(U)$ , where  $F = (f_1, f_2, f_3, \dots, f_n)^T, U = (u_1, u_2, u_3, \dots, u_n)^T$  for any orthonormal basis  $\omega = (\omega_1, \omega_2, \dots, \omega_n)^T \in \mathbb{R}^n, |\omega| = 1$ , here  $U = [\rho, \rho u, \rho v, \rho E], F = [\rho u, \rho u^2 + p, \rho v u, \rho u E + p u], [\rho v, \rho v^2 + p, \rho u v, \rho v E + p v]$ , as in equation 2.13,  $\omega_1, \omega_2, \dots, \omega_n$  are unit vectors, normal to the flux vectors  $f_1, f_2, \dots, f_n$  and Jacobian  $\mathbf{J}(F, \omega)$  is given by

$$\mathbf{J}(F, \omega) = \sum_{j=1}^n \omega_j \mathbf{J}_j(F) \quad (2.17)$$

and the matrix  $\mathbf{J}$  has  $m$  real eigenvalues  $(\lambda_1, \lambda_2, \dots, \lambda_m)$  and  $m$  linearly independent eigenvectors  $(\vec{r}_1, \vec{r}_2, \dots, \vec{r}_m)$

$$\mathbf{J}(F, \omega) = \mathbf{J}^x \omega_1 + \mathbf{J}^y \omega_2 \equiv \frac{\partial F}{\partial U} = \begin{pmatrix} \partial f_1 / \partial u_1 & \partial f_1 / \partial u_2 & \dots & \partial f_1 / \partial u_n \\ \partial f_2 / \partial u_1 & \partial f_2 / \partial u_2 & \dots & \partial f_2 / \partial u_n \\ \vdots & \vdots & \ddots & \vdots \\ \partial f_n / \partial u_1 & \partial f_n / \partial u_2 & \dots & \partial f_n / \partial u_n \end{pmatrix} \quad (2.18)$$

$$\mathbf{J}^x = \begin{pmatrix} 0 & 1 & 0 & 0 \\ \gamma \frac{u^2+v^2}{2} + \kappa - u^2 & (2-\gamma)u & -\gamma v & \gamma \\ -uv & v & u & 0 \\ u(\gamma \frac{u^2+v^2}{2} - H + \kappa) & H - \gamma u^2 & -\gamma uv & (1+\gamma)u \end{pmatrix} \quad (2.19)$$

$$\mathbf{J}^y = \begin{pmatrix} 0 & 0 & 1 & 0 \\ -uv & v & u & 0 \\ \gamma \frac{u^2+v^2}{2} + \kappa - v^2 & -\gamma u & (2-\gamma)v & \gamma \\ v(\gamma \frac{u^2+v^2}{2} - H + \kappa) & -\gamma uv & H - \gamma u^2 & (1+\gamma)v \end{pmatrix} \quad (2.20)$$

## 2.1.2 Characteristics and Eigenvalues

The characteristics in the context of a partial differential equation are a set of curves along which the Partial Differential Equation (PDE) becomes an ordinary differential equation Ordinary Differential Equation (ODE) [68]. The Jacobian matrix ( $\mathbf{J}$ ) has 4 eigenvalues and they can be found by solving  $|\mathbf{J} - \lambda \mathbf{I}| = 0$ , where  $|\dots|$  represents the determinant and  $\lambda$  are the eigenvalues of  $\mathbf{J}$ .

$$\lambda^1 = \Gamma \cdot \omega - c; \quad \lambda^2 = \Gamma \cdot \omega; \quad \lambda^3 = \Gamma \cdot \omega; \quad \lambda^4 = \Gamma \cdot \omega + c; \quad c = \sqrt{\frac{\partial p}{\partial \rho} + \frac{\partial p}{\partial e} \frac{p}{\rho^2}} \quad (2.21)$$

where,  $\Gamma = (u, v)^T$ ,  $\omega^\perp = (-\omega_2, \omega_1)^T$ ,  $\omega_1, \omega_2$  are unit vectors, normal to the flux vectors  $f_1, f_2$ ,  $c$  is the speed of sound and the eigenvectors are,

$$r^1 = \begin{pmatrix} 1 \\ u - c\omega_1 \\ v - c\omega_2 \\ H - \Gamma \cdot \omega c \end{pmatrix} \quad r^2 = \begin{pmatrix} 1 \\ u \\ v \\ H - \frac{c^2}{\gamma} \end{pmatrix} \quad r^3 = \begin{pmatrix} 0 \\ -\omega_2 \\ -\omega_1 \\ \Gamma \cdot \omega^\perp \end{pmatrix} \quad r^4 = \begin{pmatrix} 1 \\ u + c\omega_1 \\ v + c\omega_2 \\ H + \Gamma \cdot \omega c \end{pmatrix} \quad (2.22)$$

In hyperbolic systems of partial differential equations, there exists one wave for each eigenvalue [68], here in this 2D case there will be a total of 4 waves associated, in which the acoustic waves resulting from the eigenvalues  $\lambda^1, \lambda^4$  are non-linear in nature (meaning, travelling with sound speed ( $\pm c$ ) relative to the flow). If the compressible flow is supersonic, we can observe such non-linear acoustic waves in the form of shocks & expansion fans. Whereas the other two waves are a contact discontinuity across which the pressure (p) and velocity (u) are constant.

## 2.2 Equation of state (Pressure-Explicit)

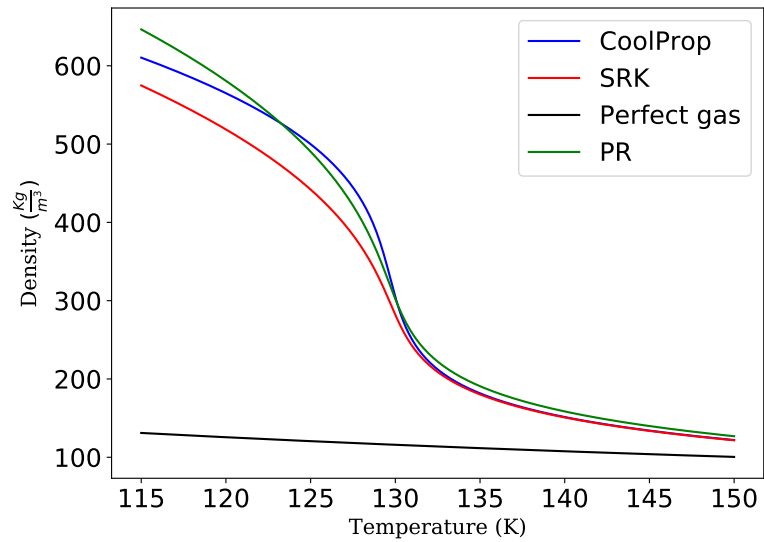
An equation of state is a mathematical expression that relates various thermodynamic state variables such as pressure, temperature, and volume/density. Some of the earliest approaches to establish such equation of states were Boyle’s Law, Ideal gas Law, Gay-Lussac’s Law, and Dalton’s law of partial pressure. Van der Waals equation of state was the first state equation to account for the intermolecular forces present in a non-ideal gas. Some of the more modern state equations are multi-parametric written in pressure explicit form. Here we present some of the more common state equations.

### 2.2.1 Cubic Equation of state

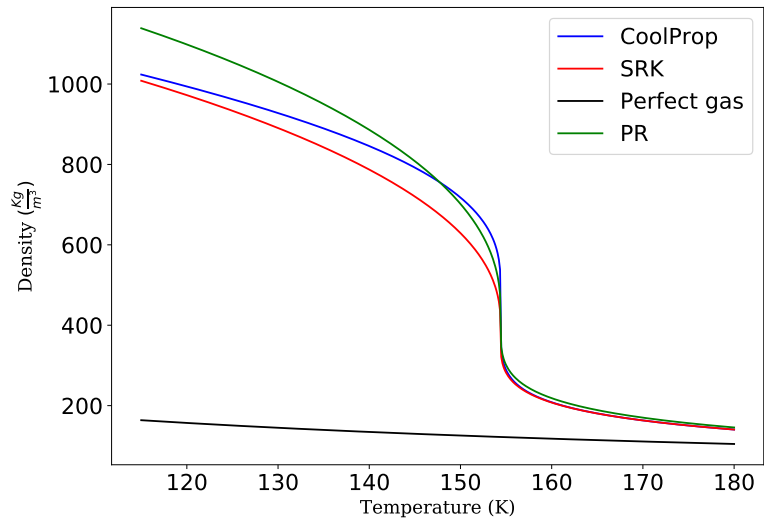
In an attempt towards accurate thermodynamic property evaluation over a wide range of temperatures and pressures, the first step was taken by Van der Waals [71] who proposed a cubic equation of state that can be applicable to both gaseous and liquid phases of a fluid. So far there are several hundred equations of states [24] that are built upon the Van der Waals EOS in order to extend the range of evaluation of real fluids.

Among those equations, the first major contributions is from Soave [58] in 1972 and then the Peng & Robinson[49] equation in 1976. Both were developed to accurately compute the thermodynamic properties for the oil and gas industries. To this day, they are the most widely used equation of states in the computational fluid dynamics software, this is largely due to their relative simplicity in terms of number of parameters involved and ability to quantify the underlying uncertainty when working with both pure components and mixtures, and also their ability to compute consistent mixture rules. Both the SRK and PR polynomial expressions have a similar form as in equation 2.23 and constants are tabulated in table 2.1

$$P = \frac{RT}{V - b} - \frac{a\alpha}{(V + \sigma b)(V + \epsilon b)} \quad (2.23)$$



(a) Density - Temperature plot of Nitrogen at 4 MPa



(b) Density - Temperature plot of Oxygen at 5 MPa

Figure 2.1: A density( $\frac{kg}{m^3}$ ) vs Temperature (K) plot for nitrogen, oxygen where the density is evaluated with different equation of states namely, [CoolProp](#)  $\bullet$ , Soave Redlich-Kwong [SRK](#), Ideal gas law, Peng-Robinson Equation of State [PR](#)

where,  $V$  is the specific molar volume,  $\alpha, \epsilon$ , are the constant parameters which differ with each equation of state, and the terms  $a, b$  are detailed in table 2.1. Even though we can compute pressure using 2.23 for a given temperature ( $T$ ), density ( $\rho$ ), to obtain density or temperature given the other two variables, one has to go through an iterative process of root-finding which is computationally inefficient, also the uncertainties associated with cubic equation of states are high [24] in comparison with experimental data or multiparametric equation of states. In terms of capabilities, when compared to SRK, RK equations of state, PR has better accuracy in calculating vapour liquid equilibrium [49].

| Parameter  | Redlich and Kwong                                    | Soave-Redlich -Kwong                                 | Peng-Robinson                                       |
|------------|--|--|---|
| $\sigma$   | 1  | 1  | $1 + \sqrt{2}$                                      |
| $\epsilon$ | 0  | 0  | $1 - \sqrt{2}$                                      |
| a          | $0.42747 \frac{R^2 T_c^2}{p_c}$                      | $0.42747 \frac{R^2 T_c^2}{p_c}$                      | $0.45724 \frac{R^2 T_c^2}{p_c}$                     |
| b          | $0.8664 \frac{RT_c}{p_c}$                            | $0.8664 \frac{RT_c}{p_c}$                            | $0.07780 \frac{RT_c}{p_c}$                          |
| k          | $0.48508 +$<br>$1.55171\omega$<br>$-0.15613\omega^2$ | $0.48508 +$<br>$1.55171\omega$<br>$-0.15613\omega^2$ | $0.37464$<br>$+1.54226\omega$<br>$-0.26992\omega^2$ |
| $\alpha$   | $\frac{1}{T_c^{0.5}}$                                | $[1 + k(1 - \sqrt{\frac{T}{T_c}})]^2$                |   |

Table 2.1: Various parameters of cubic equations of state [24], here  $\omega$  is accentric factor

## 2.2.2 Multi-parametric Equation of state

Evaluating thermodynamic properties with very low uncertainties is crucial for many industrial and scientific applications. In this section, we discuss a few such equation of states that are derived based on the fundamental relations of thermodynamics such as, internal energy as a function of entropy and volume, Gibbs energy (or) Helmholtz energy as a function of temperature and pressure, enthalpy as a function of entropy and pressure.

Among the multi-parametric high accuracy models, the Benedict-Webb-Rubin equation of state [11] or Benedict-Webb-Rubin Equation of state (BWR) is the most common form that is still being used in a number of industrial and scientific applications, thermodynamics

softwares. It was developed in 1940 by Benedict *et al.* to extend the Beattie-Bridgeman equation of state into high density regimes across various fluids. **BWR** is given as

$$p = \rho RT + \left( B_0 RT - A_0 - \frac{C_0}{T^2} \right) \rho^2 + (bRT - a) \rho^3 + a\alpha \rho^6 + \left( \frac{c\rho^3}{T^2} \right) (1 + \gamma \rho^2) e^{-\gamma \rho^2} \quad (2.24)$$

The empirical constants  $A_0, B_0, C_0, a, b, c, \alpha, \gamma$  are determined based on experimental data. This simple exponential form of representation evaluated properties in gas phase and low to medium density supercritical phases but when used for energy evaluation in the high density liquid phase and supercritical phases there was a  $\pm 10\%$  relative error [24]. There were several attempts [10, 60] to improve and extend **BWR** equation of state's capabilities into wider thermodynamic ranges across various species. Its present day version, the mBWR (modified Benedict-Webb-Rubin) comes from Jacobsen-Stewart equation of state [33], which is an advanced version of its predecessor by incorporating 32 empirical parameters making the property calculations precise and accurate.

### 2.2.3 Thermodynamic property evaluation using Pressure-Explicit Equation of State

To calculate real fluid extensive properties such as enthalpy, entropy, internal energy, we need departure functions. A departure function for a given thermodynamic property is defined as the difference between the value computed for an ideal gas and its real value (computed in its existing form) [24]. Hence, the properties like enthalpy, entropy, internal energy are evaluated relative to a base state *i.e.*, as an amount of change between two thermodynamic state spaces [24]. For example in the case of enthalpy, the change between two thermodynamic states (states 0 and n), we first compute the departure function between  $\rho_1$  and near zero density at  $T = T_0$ , then it is added to the ideal gas enthalpy change due to the temperature change from  $T_0$  to  $T_n$ , then subtract the departure function value between  $\rho_n$  and near zero density. The enthalpy of any state is calculated as,

$$\begin{aligned} H(T, \rho_n) = H^0(T^0) &+ \int_0^{\rho_n} \left[ \frac{R}{\rho_n} - \left( \frac{1}{\rho_n^2} \right) \left( \frac{\partial p}{\partial T} \right)_{\rho_n} \right] d\rho_n + \int_0^{\rho_n} \left[ \left( \frac{p}{\rho_n^2} \right) - \left( \frac{RT}{\rho_n} \right) \right] d\rho_n \\ &+ \int_{T^0}^T \left[ \frac{C_p^{pg}(T)}{T} \right] dT - R \ln(RT\rho_n) \end{aligned} \quad (2.25)$$

The entropy,  $S$ , of any thermodynamic state is calculated using,

$$S(T, \rho_n) = S^0(T^0) + \int_{T^0}^T \left[ \frac{C_p^{pg}(T)}{T} \right] dT - R \ln(RT\rho_n) + \int_0^{\rho_n} \left[ \frac{R}{\rho_n} - \left( \frac{1}{\rho_n^2} \right) \left( \frac{\partial p}{\partial T} \right)_{\rho_n} \right] d\rho \quad (2.26)$$

The speed of sound for any given fluid can be calculated as

$$c^2 = \left( \frac{\partial p}{\partial \rho} \right)_{s, X_i} = \frac{\gamma}{\rho \kappa_T} \quad (2.27)$$

Here the specific heat capacity ratio ( $\gamma$ ) is equal to  $\frac{c_p}{c_v}$  and the isothermal compressibility ( $\kappa$ ) is estimated using the volume gradient with respect to pressure as, ( $\nu$  is the volume)

$$\kappa_T = -\frac{1}{\nu} \left( \frac{\partial \nu}{\partial p} \right)_{T, X_i} \quad (2.28)$$

The heat capacity at constant volume  $C_{V,m}$  is given by,

$$C_{V,m}(T, \rho_n) = \left[ C_{p,m}^{pg}(T) - R \right] - \int_0^{\rho_n} \left[ \frac{T}{\rho_n^2} \frac{\partial^2 p}{\partial T^2} \right] d\rho_n \quad (2.29)$$

The heat capacity at constant pressure  $C_{p,m}$  calculated as,

$$C_{p,m}(T, \rho_n) = C_{V,m}(T, \rho_n) + \left[ \frac{T}{\rho_n^2} \left( \frac{\partial p}{\partial T} \right)_{\rho_n}^2 \left( \frac{\partial \rho_n}{\partial p} \right)_T \right] \quad (2.30)$$

## 2.2.4 Thermodynamics, transport properties database

The standard way of evaluating thermodynamic properties, physical properties (exceptions are surface tension, viscosity, thermal conductivity) to the highest possible accuracy is through an equation of state that is developed using Helmholtz energy formulations. It would be practically inconvenient to implement/use such formulations in a CFD software.

So, it is customary to use a database or wrapper module that evaluates the thermodynamic properties for a given physical state and composition. The most commonly used libraries are [REFPROP](#)[41], it is based on modified [BWR EOS](#) developed by [NIST](#) and its look-alike but open source version is [CoolProp](#), developed by Bell et. al[9] both libraries offer thermodynamic, transport properties for pure, pseudo-pure phase mixtures. They evaluate properties based on highly accurate Helmholtz energy formulation, and most commonly viscosity, thermal conductivity are evaluated using Chung's method [16, 17]. With the help of wrappers in various interpretable languages such as python, Matlab [CFD](#) developers can import and implement into the solvers.



# Chapter 3

## Tabulated thermodynamics for Computational fluid Dynamics

### 3.1 Tabulated Thermodynamics

State equation tabulation involves storing the thermodynamic and transport properties of the species in a library (e.g. binary file, .xml file, .json file), by doing so we move the evaluation of the [EOS](#) to a preprocessing step. Thus decoupling the [EOS](#) computational complexity from runtime performance of the solver. This approach allows for the use of accurate high-quality multiparametric backend [EOS](#) such as [BWR](#) without major runtime penalty. Tabulated approaches also provide an effective control on the thermodynamic error of the state equation. For single species fluid, two independent thermodynamic variables are needed to fully define the local thermodynamic state in a single phase. Classically, the choice of the thermodynamic tuple has been pressure and temperature given the convenience of computing the isothermal and isobaric conditions. For CFD simulations, this approach is sub-optimal since the known transported quantities (in a conservative form) are  $\rho$  and  $e$ ; thus making these the ideal basis for any thermodynamic table. Once the corresponding position in the table is found, all thermodynamic and transport properties may be interpolated or computed. Here, we consider three different tabulation approaches:

- Homogeneous of logarithmic tabulation
- Block structured Adaptive Mesh Refinement ([AMR](#))
- Adaptive Mesh Refinement with Bezier patch fitting

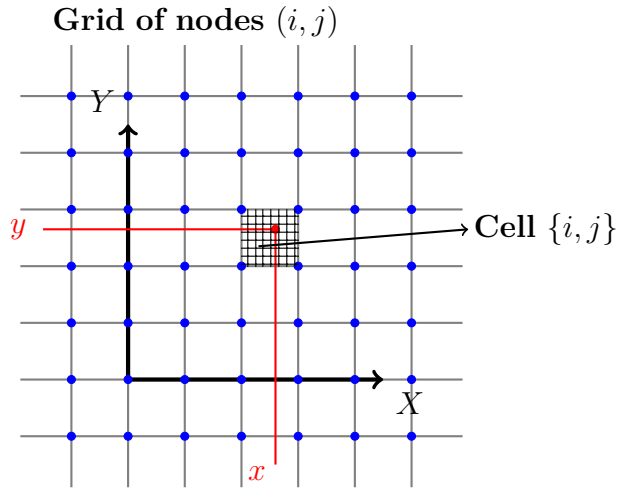


Figure 3.1: A typical cell in a 2D lookup table, the red dot indicating the unknown point

### 3.1.1 Previous works

The quest for calculating thermodynamic properties using interpolation approaches started in 1977, Theodore E. Fessler[22] worked on a "double-lagrangian" interpolation method that evaluates thermodynamic properties at intermediary points  $(x_i, y_i)$  to the tabulated nodes  $a_1, a_2, a_3$ , with the help of approximated functions  $(F(a_1), F(a_2), F(a_3))$ . This tabulation method has been explained and implemented in section 3.2.

Kunick [39] developed a "spline-based table look-up method (SBTL)" and also successfully tested it in a CFD solver with water and steam as working fluids, the SBTL method combines global bi-quadratic spline interpolation, a combination of linear transformation functions. In a project of International Association for the Properties of Water and Steam (IAPWS) the SBTL method has been applied to the industrial formulation of IAPWS-97. SBTL property functions for water and steam have been generated for the independent variables specific volume ( $\nu$ ) and internal energy ( $e$ ).

However there are some shortcomings in this approach, the nodes are often clustered to consider the nonlinear behaviour of the fluid property function, which leads to computationally intensive cell search algorithms and also most frequently applied property functions are often calculated from inverse functions, rather than from an explicit forward function. [39]

As the requirement of an adaptive Cartesian mesh is obvious in the transcritical regions of the look-up table, many researchers started to develop such irregular grid spacing

meshes. Even though there were many readily available adaptive meshing procedures, they are primarily used in the field of numerical analysis, computational mathematics, the first approach to apply an adaptive Cartesian mesh for complex fluid computations is done by Xia et al.[73]. The thermodynamic properties and their pertinent derivatives are evaluated by means of an adaptive Cartesian mesh in the thermodynamic plane that provides user-specified accuracy over any selected domain. After comparing with the REFPROP database, the properties are not only accurate but also their reconstructions are approximately two orders of magnitude faster than property evaluations from the original database(REFPROP).

Carpenter[15] worked on an adaptive tabulation scheme for multi-phase equations of state where in the computational efficiency is provided through the use of a quad-tree representation. Using both rectangular and triangular interpolation regions resulting in a accurate description of phase boundaries.

Zhiqi et. al[42] constructed an adaptive tabular thermodynamic representation with triangular elements, which is an efficient method to reconstruct the phase boundaries, thermodynamic properties of real fluids. It can provide user-defined accuracy over a selected thermodynamic plane with fewer nodes than a simple homogeneous tabulation. A generic function approximation algorithm for real valued functions in two independent variables is developed by Luke [19] where in a method for evaluating the approximate surface is generated using a piecewise  $C^1$  - continuous cubic Bezier surface. The significance of this approach is observed in terms of the number of patches required to fit the supercritical region for a given accuracy of 0.1% (in reference to the REFPROP database) which is 1,968 whereas the isotropic tabular formulation of Xia et al[73] required 225,121 patches to fit. Though, Xia et al[73] reported similar speedup of performance improvements over REFPROP, the an-isotropic refinement used by Luke et al. [19]’s tabular fitting technique improved the efficiency of tabulation in terms of storage.

## 3.2 Homogeneous or Logarithmic tabulation

Homogeneous tabulation is the simplest form of storing data in a multidimensional array covering the entire thermodynamic domain with an equal spacing or alternatively under a known mapping function like a logarithmic distribution, so that the indexes map to the structured data points as shown in figure 3.1. The advantage of using such an approach is that the lookup time complexity is minimal as it is a matter of calculating the normalized x, y values and mapping them to the index for interpolation. But, this results in very large table sizes as the grid size is not adapted to the magnitude of the local error.

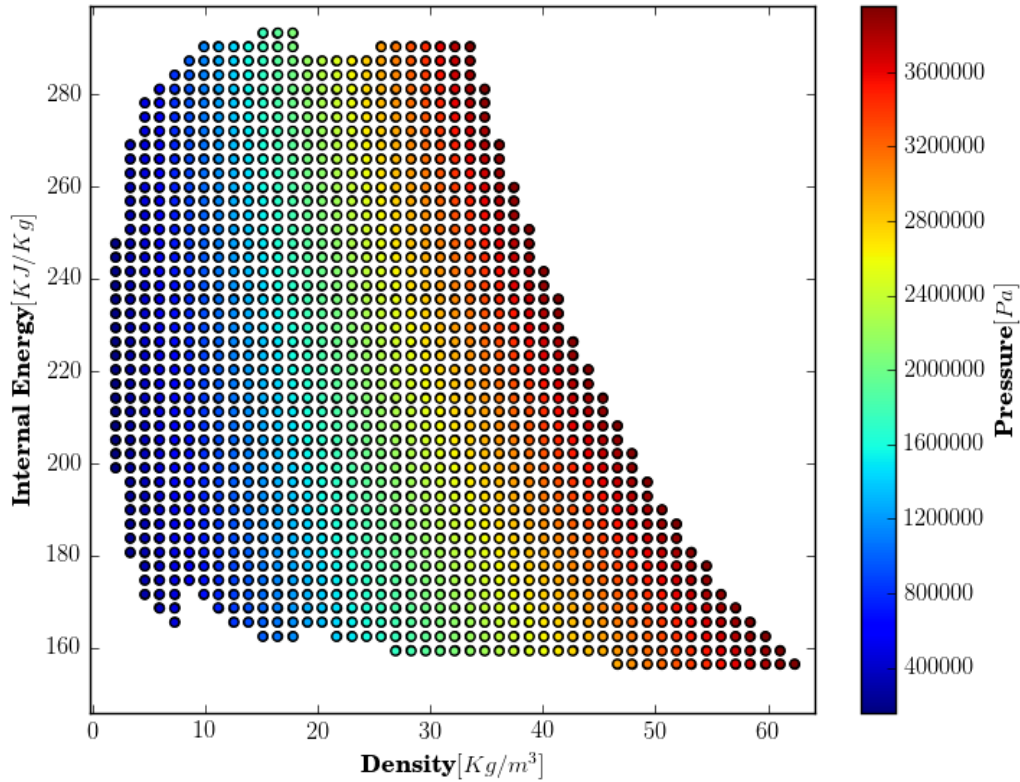


Figure 3.2: Uniform spaced EOS table for oxygen in internal energy ( $E$ ), density ( $\rho$ ) space.

The minimal accuracy is in the region in which the function varies most rapidly so that slowly varying regions must be over-refined to accommodate high-gradient regions. Tabulating in density/energy space, as shown in figure 3.2, allows for a simple lookup from the transport equations but simultaneously the corner regions of the table contain pressure and temperature values that are far outside the region of interest for most CFD problems. Therefore, typical boundaries of isobaric conditions of the simulation are mapped to the energy/density space.

### 3.3 Block structured Adaptive Mesh refinement (AMR)

In order to achieve an uniform accuracy of tabulated thermodynamic properties (i.e., when the error computed with reference to a [NIST](#) database), we have to balance or adjust the grid spacing on the basis of local property gradient throughout the thermodynamic domain space. Such adaptable grid spacing or "mesh refining" methods fall under the category of unstructured grids. Block structured adaptive mesh refinement is one such method, where the grid spacing/cell size is based on a hierarchical level of resolution that is defined for a particular block (a structured group of cells) within which the cell lies.

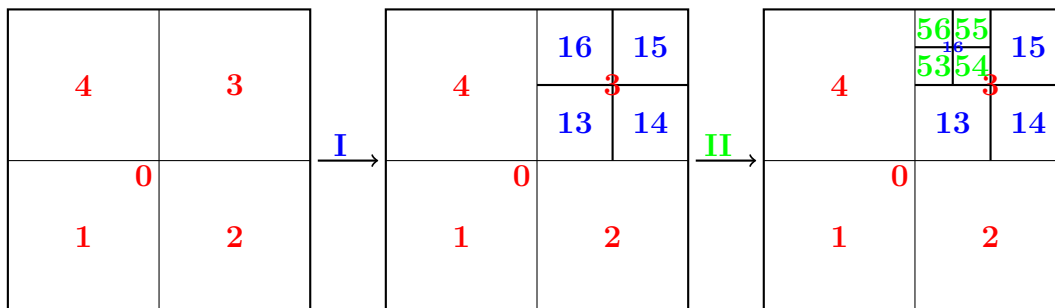


Figure 3.3: Block structured Adaptive Mesh refinement

The criteria on which a given block gets a new hierarchical level of resolution primarily depends on the accuracy with which we are able to compute the thermodynamic parameters. Blocks associated with a non-linear change in property values such as near-critical region of a phase diagram will have a finer grid spacing than the regions where the property gradient is much smoother or linear.

#### 3.3.1 AMR Table Generation

The adaptive mesh refinement tabulation is based on Cartesian adaptive mesh structure. A rectangular x-y region defined by the table limits and these x, y variables can be Pressure, Temperature, density or internal energy. During initialization, the unknown thermodynamic properties are evaluated at each of the rectangle corners using [NIST](#) database and an approximate reconstruction method (interpolation) is used to evaluate these properties at pre-selected equidistant points. For simplicity, we chose bivariate interpolation to evaluate the properties at those selected 25 equidistant points for each cell created due to subdivision. We start by transforming the rectilinear domain of interest (i.e.,  $\chi$ ) into a

square coordinate system ( $\kappa$ ) and the condition for subdivision is defined based on relative error( $\Phi$ ) computed with reference to [NIST](#) database as,

$$\Phi = \frac{\chi_{Interpolated} - \chi_{NIST}}{\chi_{NIST}} \quad (3.1)$$

---

**Algorithm 1:** Block Structured Adaptive refinement based Tabular [EOS](#) Generation

---

```

 $\epsilon \leftarrow$  User defined Error tolerance;
 $\eta \leftarrow$  Table maximum resolution limit;
 $[(min_1, max_1), (min_2, max_2)] \leftarrow$  State space( $\chi$ ) range (P-T) or ( $\rho$ -e);
 $f : \chi \rightarrow \kappa$  Projective transform( $f$ ) of state space( $\chi$ ) into square domain( $\kappa$ );
Evaluate thermodynamic properties using NIST,  $\forall v \in [(min_1, max_1), (min_2, max_2)]$ ;
Initialize the Quadtree Root node with transformed state space;
Function Subdivide(Rootnode):
    Add new child nodes with properties evaluated, to the list;
    for child in chldnodes : do
        for point  $\leftarrow$  1 to 25 points do
            Interpolate thermodynamic properties using 4 neighbour nodes;
            Relative Error  $\Phi \leftarrow$  Computed using 3.1 ;
            Check for thermodynamic consistency using 3.4;
            Append error to a list;
        end for
        if  $max(Errors(\Phi)) \geq \epsilon$  &  $Quadtree.depth < \eta$  then
            Subdivide(child);
        else
        end for
    End Subdivide;
    Prune(Rootnode);
    Traverse(Rootnode);
    Save Rootnode and its leaves in a list;
return Table;

```

---

The reconstructed (interpolated) properties are validated with the reference data obtained from [NIST](#) database or any backend [EOS](#) to quantify the interpolation error. If the error is more than the user-specified tolerance, the domain is further divided into four equal regions (squares), again they will be validated by comparing with the exact [NIST](#) database values. With such repetitive subdivisions, a majority of flatter(linear) regions

of the mathematical expression will satisfy the desired accuracy within few subdivisions whereas the areas of sharper change such as near-critical region will be subdivided further. Storing this data in a dynamically spaced quadtree format can able quick lookup that can be comparable to the speed of equally of spaced tables.

### 3.3.2 Lookup Approach

The usual query/search in quadtrees follows either top-down or bottom-down approach like starting from the root node and isolate the child within which the point belongs to or vice versa. This has a time complexity of  $O(\text{Log } n)$  where 'n' is the depth of tree. As an alternative approach, we employ a fast lookup algorithm that requires additional memory to store the indices in tree data-structure, and the efficiency doesn't depend much on the depth of the tree. It is constructed by the following steps.

A uniform index array based on quadtree subdivision covering the entire computational domain is generated as per the pre-defined maximum refinement level. For example, at a given maximum refinement level of 3, the uniform index table is shown in the figure 3.5. If the maximum refinement level is  $n+1$  (the refinement, level of the single rectangular cell that cover the entire thermodynamic region of interest is 0) then there will be  $4^n$  number of individual patches (in this case rectangular cells), the integer number stored in the cell indicates the index of that patch.

```
1 uni_old = np.zeros((1,1))
2 for n in range(QuadTree.maxdepth):
3     uni_new = np.zeros((2**(n+1), 2**(n+1)))
4     for i in range(2**n):
5         for j in range(2**n):
6             uni_new[2*i][2*j] = int(uni_old[i][j]*4 + 1)
7             uni_new[(2*i)+1][2*j] = int(uni_old[i][j]*4 + 2)
8             uni_new[(2*i)+1][(2*j)+1] = int(uni_old[i][j]*4+3)
9             uni_new[2*i][(2*j)+1] = int(uni_old[i][j]*4 + 4)
10    uni_old = uni_new
```

Listing 3.1: Generating an uniform array index

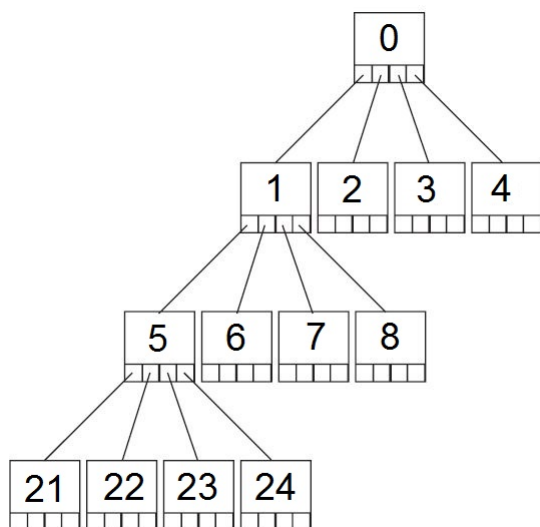


Figure 3.4: Sample quadtree subdivision

|    |    |    |    |    |    |    |    |
|----|----|----|----|----|----|----|----|
| 84 | 83 | 80 | 79 | 68 | 67 | 64 | 63 |
| 81 | 82 | 77 | 78 | 65 | 66 | 61 | 62 |
| 72 | 71 | 76 | 75 | 56 | 55 | 60 | 59 |
| 69 | 70 | 73 | 74 | 53 | 54 | 57 | 58 |
| 36 | 35 | 32 | 31 | 52 | 51 | 48 | 47 |
| 33 | 34 | 29 | 30 | 49 | 50 | 45 | 46 |
| 24 | 23 | 28 | 27 | 40 | 39 | 44 | 43 |
| 21 | 22 | 25 | 26 | 37 | 38 | 41 | 42 |

Figure 3.5: Uniform index for a maximum refinement level of 3

|    |    |    |    |    |    |    |  |
|----|----|----|----|----|----|----|--|
| 20 |    | 19 |    | 16 |    | 15 |  |
| 17 |    | 18 |    | 13 |    | 14 |  |
| 35 | 36 | 32 | 31 | 12 |    | 11 |  |
| 33 | 34 | 29 | 30 | 12 |    | 11 |  |
| 24 | 23 | 28 | 27 | 40 | 39 | 10 |  |
| 21 | 22 | 25 | 26 | 37 | 38 | 10 |  |

Figure 3.6: Overlapping index for uniform index at refinement levels  $n = 0, 1, 2, 3$

An overlapping mapped index array is also generated which contains the child node index in the place of all further sub divisions pertaining to that child if there was a refine-



ment as illustrated in figure 5. The cell within which a point  $(T_x, P_x)$  is located can be determined at an algorithmic complexity  $O(1)$ , directly without searching the tree (which is of algorithmic complexity  $O(n)$  where  $n$  being the refinement level), by using the mapping index array. Furthermore, the storage required for the uniform index table is small for this generation computers, for example, if the maximum refinement level of the adaptive table is 10, and an integer requires 4 B of memory, the uniform index table requires  $(2^9 \times 2^9 \times 4) / (1024^2) MB = 1 MB$ .

---

**Algorithm 2:** Block Structured Adaptive refinement based Tabular EOS Lookup

---

**Input:** (Pressure(P), Temperature(T)) or (Density( $\rho$ ), Internal-Energy)  
**Output:** List of thermodynamic properties  
 $\partial\chi_1, \partial\chi_2 \leftarrow$  Calculate the grid resolution based on maximum depth of tree;  
 $f : \chi \rightarrow \kappa$  Projective transform( $f$ ) of state space( $\chi$ ) into square domain( $\kappa$ );  
 Compute the x-index, y-index based on the ratio of x-value, y-value to grid spacing;  
 $idx \leftarrow \text{floor}\left(\frac{\chi_1}{\partial\chi_1}\right)$ ;  
 $idy \leftarrow \text{floor}\left(\frac{\chi_2}{\partial\chi_2}\right)$ ;  
 $v \leftarrow \text{Uniform Index}[idy][idx] \quad \triangleright O(1)$ ;  
 $u \leftarrow \text{Overlapping Index}[v] \quad \triangleright O(1)$ ;  
 $w \leftarrow \text{np.where(Quadtree list == u)} \quad \triangleright O(1)$ ;  
 $Data \leftarrow \text{Quadtreedata}[w] \quad \triangleright O(1)$ ;  
 $P_n \leftarrow Data[0] \quad \triangleright$  The square nodes in which the point lies;  
 $D_n \leftarrow Data[1:] \quad \triangleright$  Access thermodynamic property data stored in those nodes;  
 $P \leftarrow \text{Interpolation}(P_n, D_n)$ ;  
**return**  $P$ ;

---

### 3.3.3 AMR Storage

Once we subdivide the tree based on the underlying non-linear equation of state as described in algorithm 1, we traverse the tree top down and update the leaves (nodes) of the quadtree. Then for each element in leaves (Quadtree.nodes) we append the node's stored properties into a list file("Quadtree-leaves"), also as the data is unstructured we keep track of their indices into another file ("Quadtree index"). A uniform index array is constructed as described in 3.3.2 based on the table's observed maximum tree depth and written into a file ("Uniform Index"). The final task is to construct an overlapping index with the help of previously constructed quadtree index file and the uniform index file. The overlapping index superposes the uniform index in such a way that for any given unknown coordinate the overlapping index overrides its uniform index except, in case of child nodes

where they both are equal. Finally, the projective transform module is also stored so that the same mapping will be used during the lookup to convert the thermodynamic state space variables  $(\chi_1, \chi_2)$  into a square coordinates  $(\kappa_1, \kappa_2)$ .

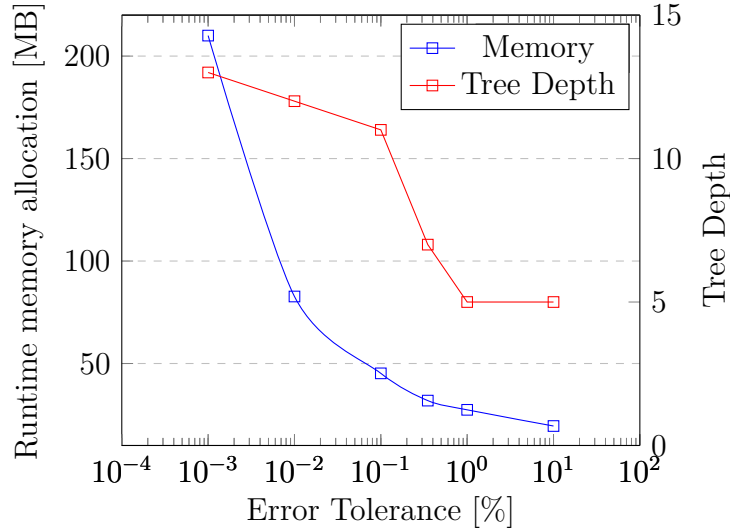


Figure 3.7: Memory allocation of Block structured AMR tabular EOS during runtime in Random Access Memory, for the physical conditions range; Pressure: 1.0 MPa - 10.0 MPa, Temperature: 100K - 900K, Oxygen. Here the error is calculated in reference to underlying backend EOS

In terms of size of files that need to be loaded into the Random access memory (RAM) during the simulation runtime, it varies corresponding to the accuracy and range within which are constructing the table. Figure 3.7 shows the runtime memory allocation with respect to error tolerance for a fixed range of physical conditions i.e., Oxygen in pressure (1.0 MPa - 10.0 MPa) and temperature (100K - 900K). Also the table size is proportional to the maximum resolution to which the subdivision was allowed.

### 3.3.4 Visualization of the tabular thermodynamics

To visualize the generated tabular EOS, we recursively plot the cells, to provide a easier understanding readers can compare the below plots to the phase diagram (figure 1.1). The below (P, T) tables are for oxygen, with decreasing order of error tolerances (0.1%, 0.01%, 0.001%) and  $(\rho, e)$  table is also presented (figure 3.11). The evaluation ranges

are  $[P_{min}, T_{min}] = [0.02E06 \text{ Pa}, 85 \text{ K}]$ ,  $[P_{max}, T_{max}] = [10E06 \text{ Pa}, 600 \text{ K}]$ . Whereas in the case of  $(\rho, e)$  table, boundaries are  $[\rho_{min}, e_{min}] = [0.14 \frac{Kg}{m^3}, 50000 \frac{J}{Kg}]$  and  $[\rho_{max}, e_{max}] = [130.20 \frac{Kg}{m^3}, 300000 \frac{J}{Kg}]$

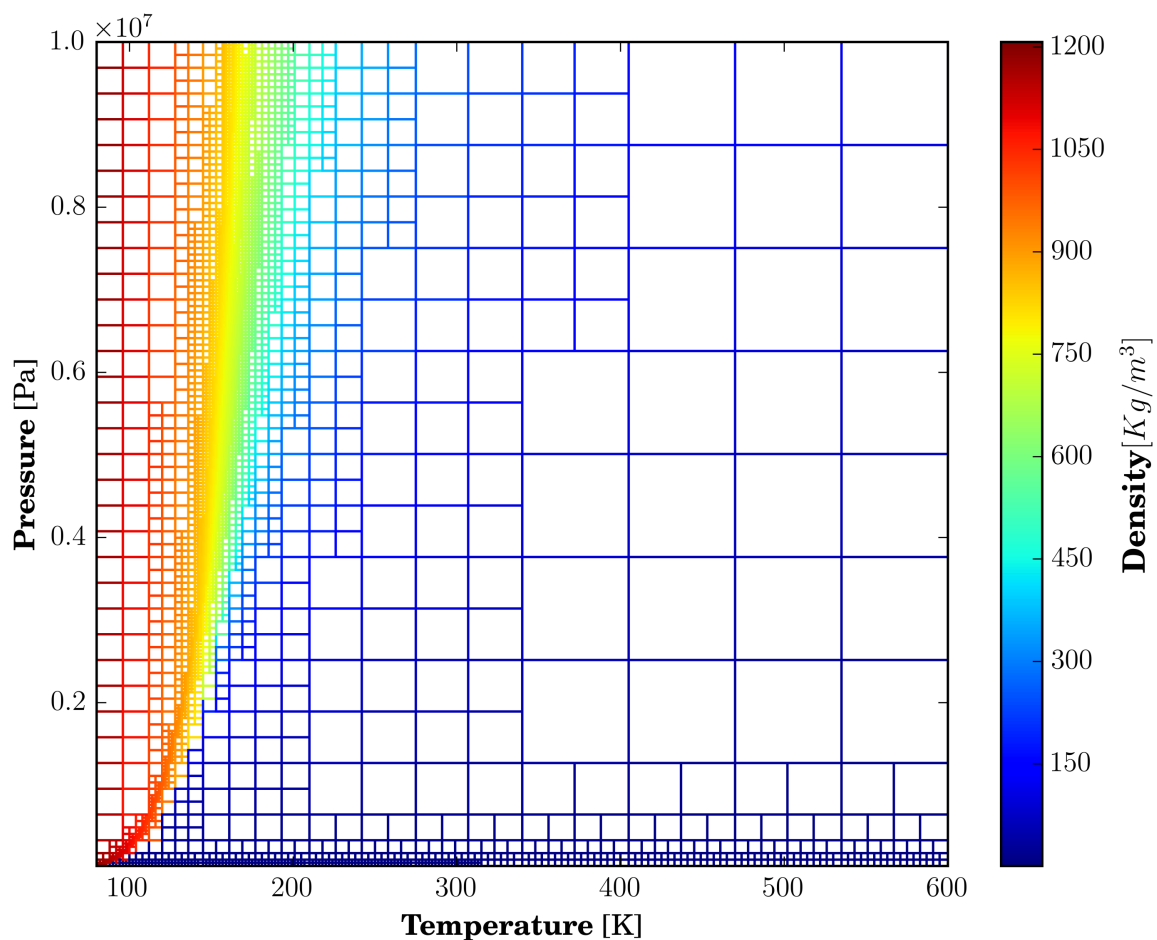


Figure 3.8: Block structured AMR Pressure(P) -Temperature(T) lookup table for oxygen with 0.1% error tolerance compared to NIST's REFPROP database.

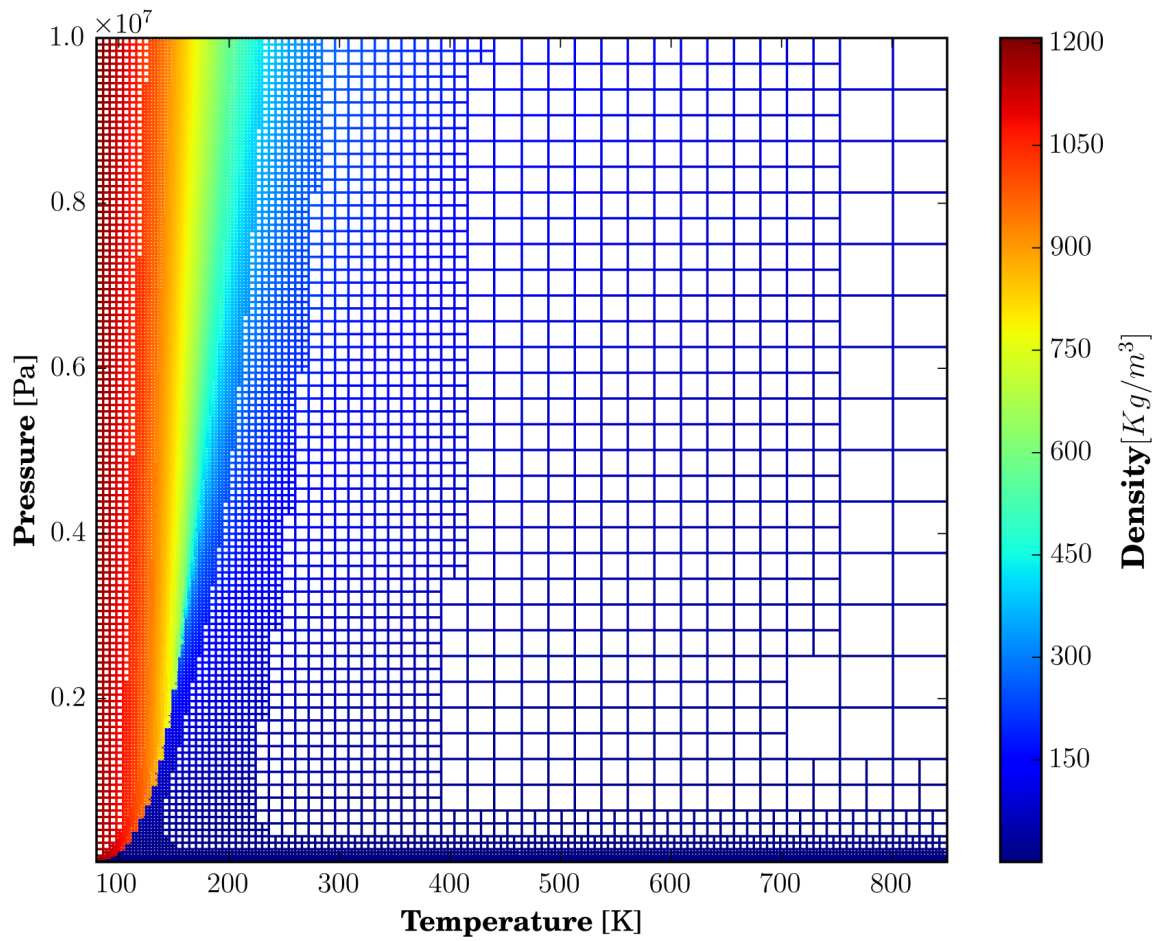


Figure 3.9: Block structured AMR Pressure(P) -Temperature(T) lookup table for oxygen with 0.01% error tolerance compared to NIST's REFPROP database.

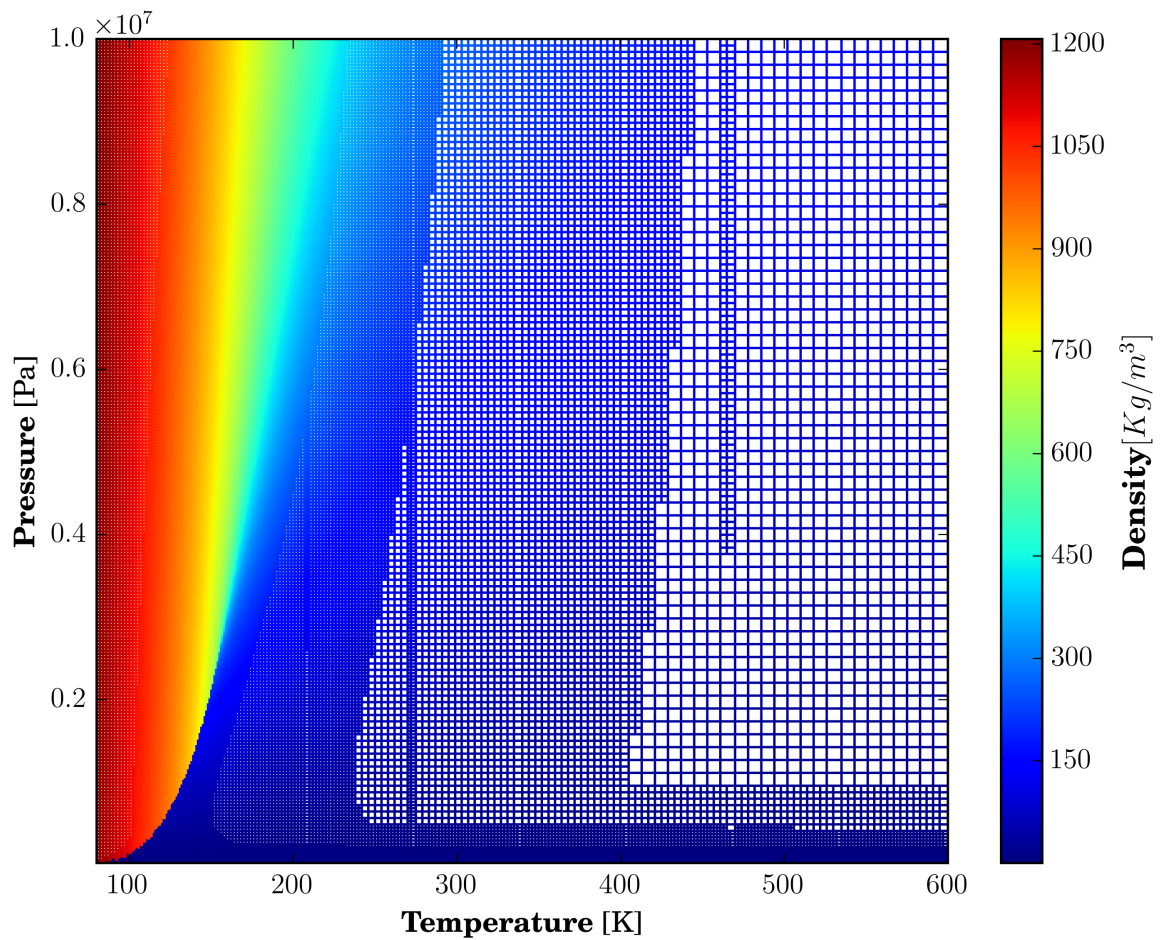


Figure 3.10: Block structured AMR Pressure(P) -Temperature(T) lookup table for oxygen with 0.001% error tolerance compared to NIST's REFPROP database.

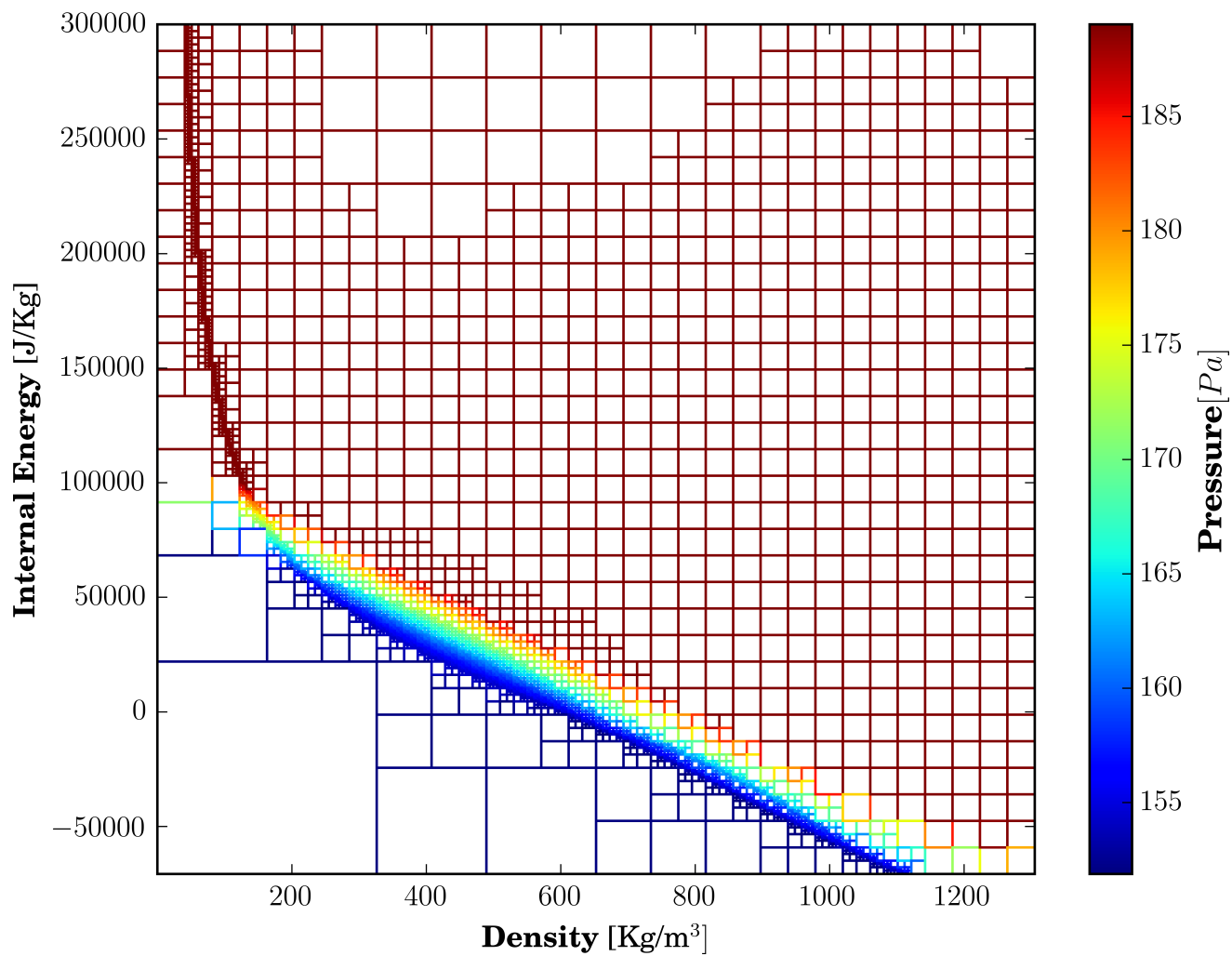


Figure 3.11: Block structured AMR lookup table for oxygen with 0.01% accuracy compared to NIST's REFPROP database., generated in  $(\rho, e)$  state space

### 3.4 AMR with Bezier patch fitting tabulation [31]

This method is a work done by Matthew Yao[31], for the block structured AMR table, the underlying criteria for refinement is based on the multivariate approximation. Rather than tabulating solely on the basis of bilinear interpolation error we can implement Bezier surfaces/patches that are approximated functions. This function of two independent variables is [19], here,  $B_i^3(u)$  are the third-degree (fourth-order) Bernstein polynomials,  $b_{ij}$  are the control points.

$$F(u, v) = \sum_{i=0}^3 \sum_{j=0}^3 B_i^3(u) B_j^3(v) b_{ij} \quad (3.2)$$

$$B_i^3(u) = \frac{3!}{i!(3-i)!} (1-u)^{3-i} u^i \quad (3.3)$$

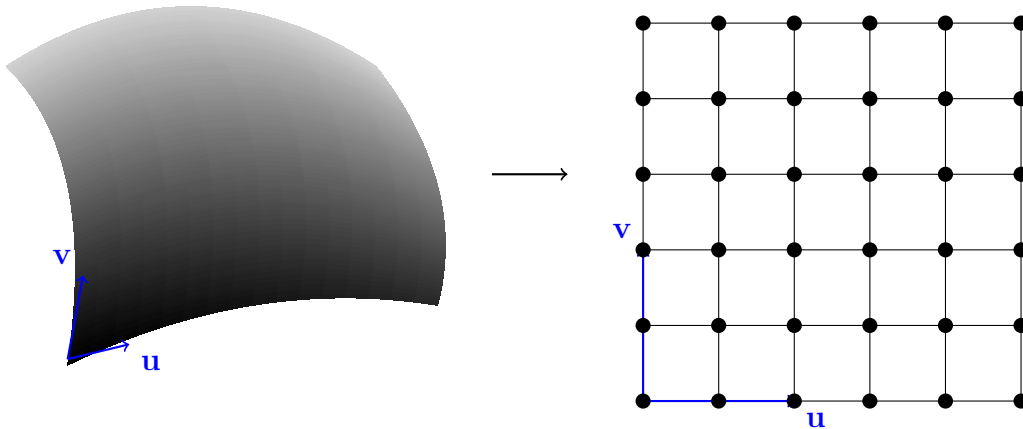


Figure 3.12: Sample Bezier patch being adapted into local grid

Using the linear combination of these  $b_{ij}$  control points, the shape of the surface is determined. The surface is generated by recursively subdividing the domain until the resulting Bezier patches are capable of reconstructing the original value to a specified error threshold or if the maximum refinement level is reached. The table generation process is similar to the previously described Block structured AMR, we use a quad tree to recursively refine the thermodynamic domain by dividing into four patches and evaluate the properties within each patch.

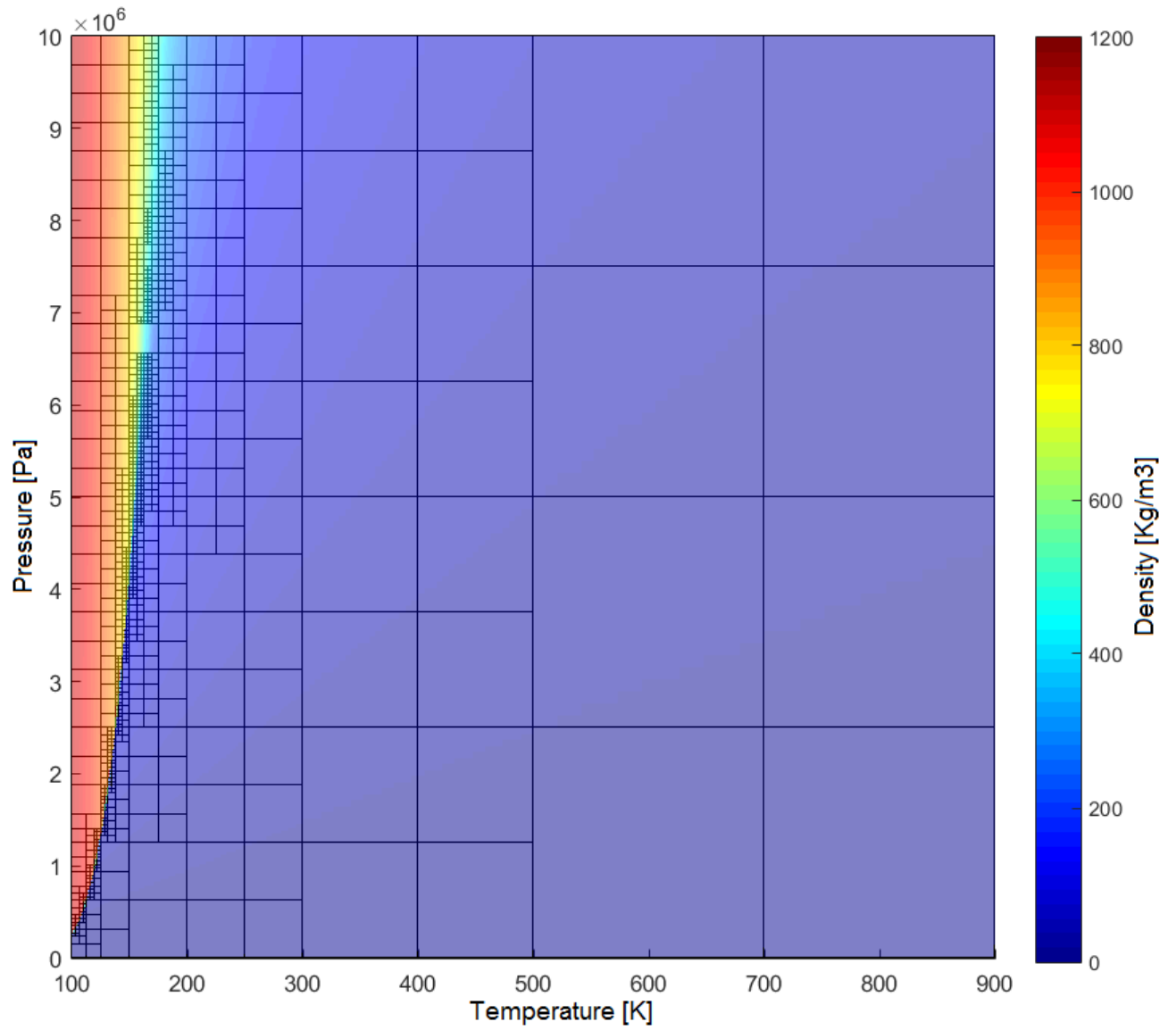


Figure 3.13: Bezier patch based Tabulated EoS for Density( $\rho$ ) adaptive refinement using quadtree, reproduced with permission from Matthew Yao[31]



### 3.5 Thermodynamic consistency

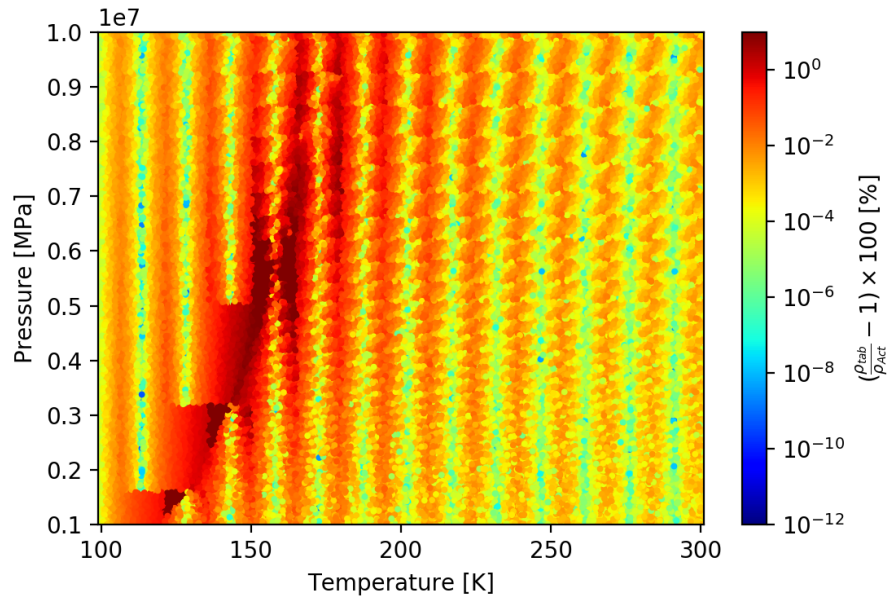
A valid [EOS](#) must satisfy the thermodynamic condition of consistency. Maintaining thermodynamic consistency involves satisfying the Maxwell relations [\[61\]](#) and also thermodynamic properties such as pressure(P), density( $\rho$ ), internal energy(e) must confine to their definitions in these derivatives. A tabular [EOS](#) may exhibit wide range of behaviour over the range of temperature and density if it is not consistent. Such behaviour manifests itself as sharp discontinuities in thermodynamic property evaluation near phase boundaries. In [CFD](#) codes, it is important to have a tabular interpolation scheme that can produce consistent thermodynamic properties. Even insignificant inconsistencies of the order of  $\approx 10^{-5}$  can pose difficulties for the implicit solution of underlying governing equations. In order to be able to construct accurate fluxes, we must be able to know how to evaluate physically realistic values of several dimensionless quantities, thermodynamic derivatives. For example, if we express internal energy E as a function of entropy(S) and specific volume(V). The thermodynamic property definitions of temperature(T) and pressure(P) are

$$T = -\left.\frac{\partial E}{\partial S}\right|_V, P = -\left.\frac{\partial E}{\partial V}\right|_S \quad (3.4)$$

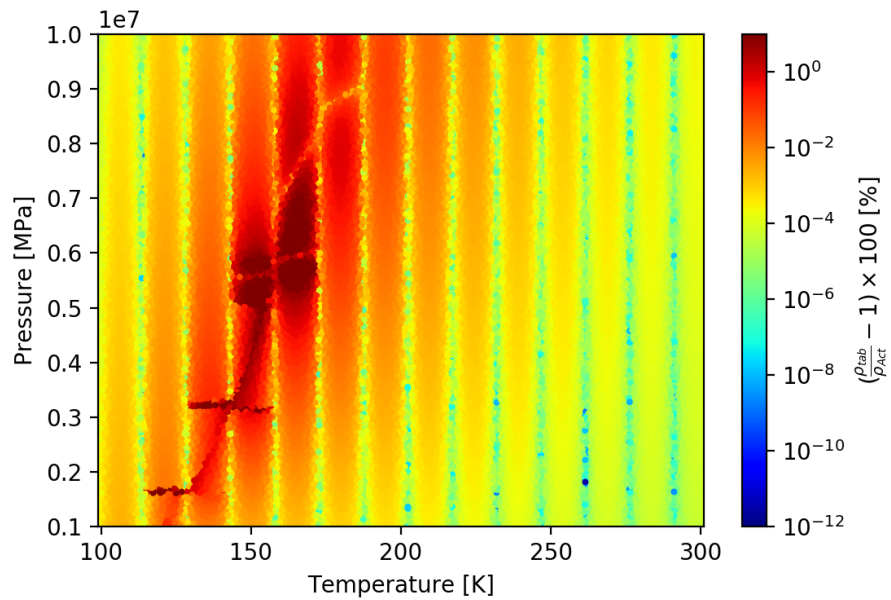
This is one of the thermodynamic consistency condition. With the bilinear interpolation, both the density and its first order derivatives vary linearly over each local rectangular cell. Nevertheless, as the thermodynamic properties stored on the mesh nodes are themselves obtained from a consistent thermodynamic database such as [NIST](#), [CoolProp](#)), in a fine tabulation such intrinsic inconsistencies are going to be small. [\[73\]](#)

### 3.6 Error Estimates & Verification

To verify and understand the accuracy of the tabulation methods, we uniformly random sample the entire tabulation domain (T-P,  $\rho$ -E) and evaluate the error in density evaluation. Starting with the homogeneous tabulation and considering two interpolation schemes i.e., bilinear and bicubic the contour error plots are shown in figure [3.14a](#), [3.14b](#). when compared both, bicubic interpolation performed well in terms of maintaining accuracy. Except in the vapour-liquid curve and the supercritical state. This is attributed to the usage of derivatives across the interpolating nodes and thereby making bicubic interpolation  $C^1$  continuous, which is the feature lacking in bilinear method as can be seen in the figure [3.14a](#) the erroneous regions form patch like shape in between the grids compared to much smoother transition observed in the latter case.



(a) Tabulation with bilinear interpolation



(b) Tabulation with bicubic interpolation

Figure 3.14: Homogeneous Tabulation Relative Error[%] in Density estimates

The T-P tabular grid is sampled into 10000x10000 linear-linear grid (figure 3.15), log-linear (figure 3.17). In each patch 100 points are sampled and then the maximum relative error in density is computed. As seen from the relative error contour plot, the maximum error is along the phase change line or where the density changes rapidly. Except for that region the uniform accuracy is maintained as a result of adaptive subdivision. The highest error points lies close to the critical point. The number of patches required to capture the thermodynamics is different in three approaches (Homogeneous, Adaptive, Bezier patch) The highest being the homogeneous due to its uniform spacing, it will take enormous number of patches to get the error close to 0.01%, although Bezier patch has less number of patches it can only accommodate one property at a time, which means it requires individual table for each property. The Block based AMR has moderate number of patches that are not oversized but can tabulate all thermodynamic properties.

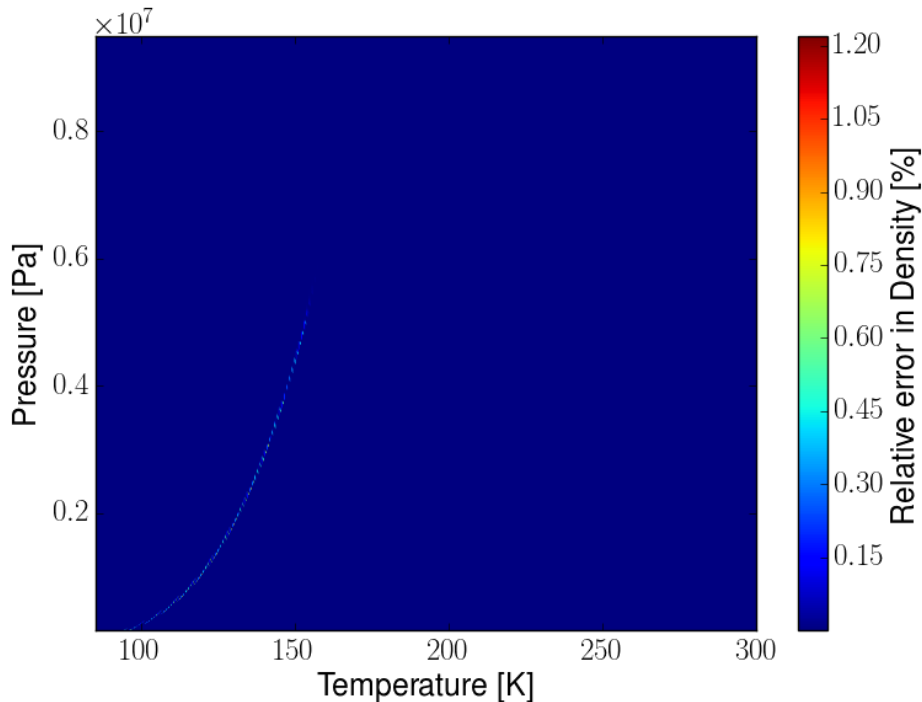


Figure 3.15: Relative error in density for the Block structured adaptive tabulation

The comparison of various tabulation approaches demands a quantitative thermodynamic error metric. As the exactness thermodynamic properties remain a subject of scientific relevance, the current study evaluates the error relative to the NIST database. These

data points represent the highest fidelity data set for the computation of the thermodynamic and thermophysical properties and constructed from an ensemble of experimental data supplemented with Modified Benedict-Webb-Rubin curve fitting.

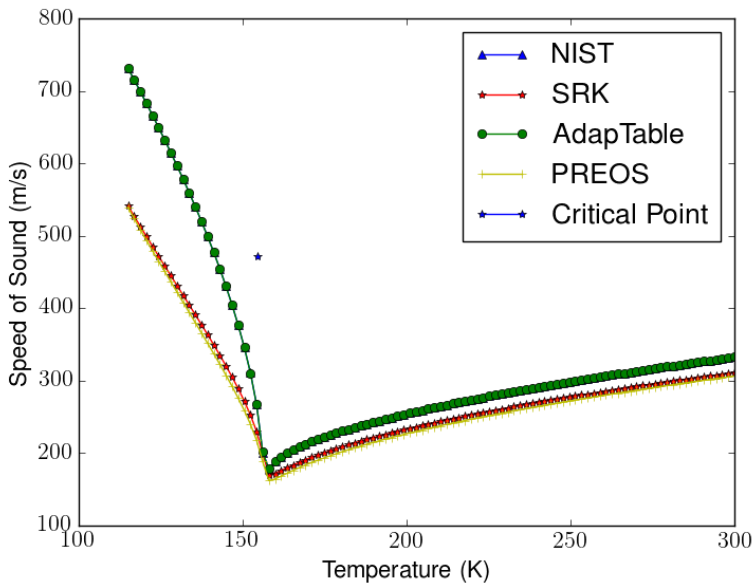


Figure 3.16: Speed of sound for Oxygen computed with REFPROP, SRK, PR, tabulated AMR EOS (AdapTable)

Table 3.1: Error estimates

| Grid type                       | Block based AMR<br>(0.0001 tol.) | Bezier Patch<br>AMR<br>(0.0001 tol.) |
|---------------------------------|----------------------------------|--------------------------------------|
| Total patches                   | 12648                            | 1120                                 |
| % patches with rho error >0.001 | 0.101                            | -                                    |
| Maximum rho error %             | 1.2                              | -                                    |

To evaluate different types of equation of states, we employ few statistical techniques such as Average Percent Relative Error (ARE%), calculated as a reference measure

which is defined below is a measure of the bias of the correlation; a value of zero indicates a random of the measured values around the correlation.

$$ARE\% = \frac{100}{N_d} \sum_{i=1}^{N_d} \frac{(\rho_i^{exp.} - \rho_i^{calc.})}{\rho_i^{exp.}} \quad (3.5)$$

The AARE%, is the arithmetic average of the absolute values of the relative errors; is an indication of the accuracy of the correlation.

$$AARE\% = \frac{100}{N_d} \sum_{i=1}^{N_d} \frac{|\rho_i^{exp.} - \rho_i^{calc.}|}{\rho_i^{exp.}} \quad (3.6)$$

The  $R^2$ , is the correlation coefficient; is a measure of the precision of fit of the data. If data are perfectly correlated, then  $R^2 = 1$ . A small value of AARE% and  $R^2$  close value to one (simultaneously) denote a good correlation based on good data.

Another parameter, Sum of Absolute of Residual (SAR) shows the reliability of correlation for higher order data points.

$$SAR\% = \sum_{i=1}^{N_d} |\rho_i^{exp.} - \rho_i^{calc.}| \quad (3.7)$$

The statistical parameters of the well known EoSs are listed in table 3.2

Table 3.2: Error metrics comparison with NIST

| Equation of State                               | AARE  | ARE    | SAR [Kg/m <sup>3</sup> ] | R <sup>2</sup> |
|---|-------|--------|--------------------------|----------------|
| Peng-Robinson <a href="#">PR</a>                | 3.015 | 0.993  | 70.600                   | 0.991          |
| Redlich-Kwong <a href="#">RK</a>                | 4.409 | -0.026 | 89.816                   | 0.987          |
| <a href="#">AMR</a> Tabular <a href="#">EOS</a> | 1.022 | 0.0021 | 91.232                   | 0.995          |
| Soave-Redlich-Kwong <a href="#">SRK</a>         | 6.388 | 6.381  | 131.009                  | 0.990          |

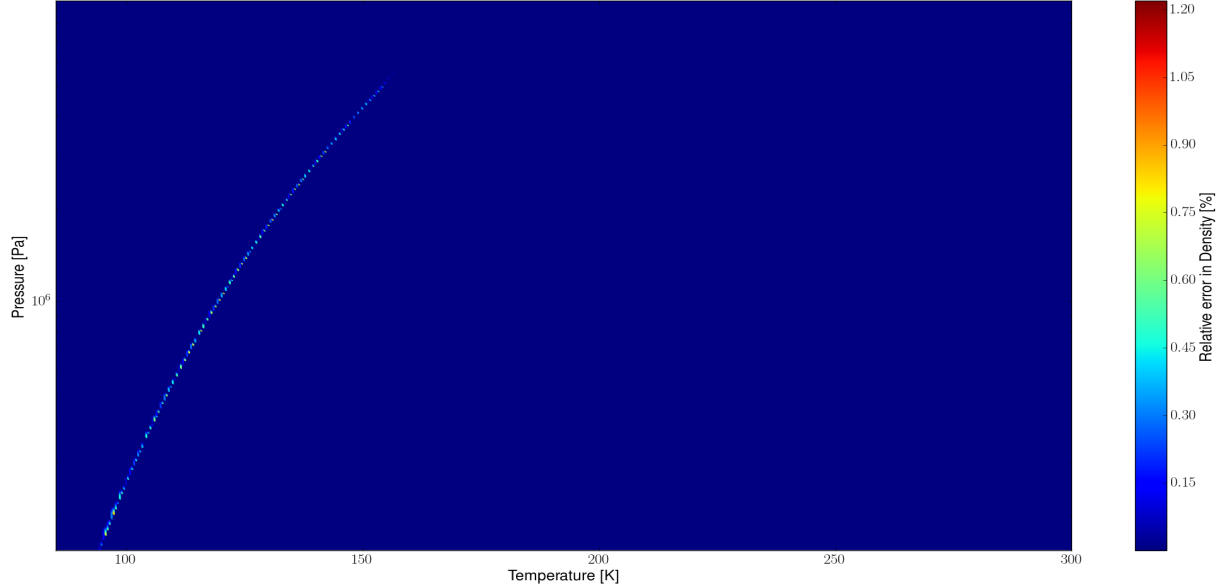
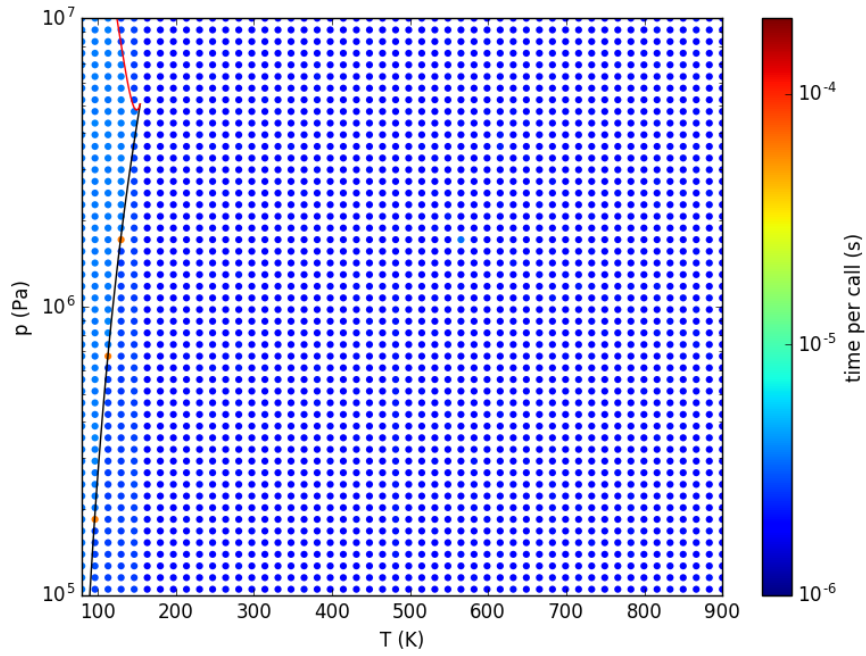


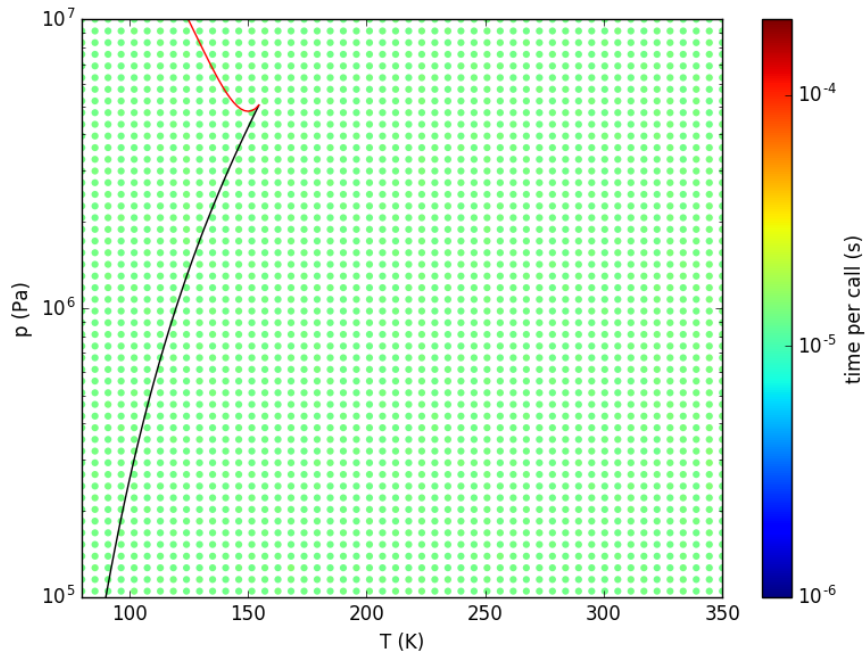
Figure 3.17: Error Validation (Log-Linear plot)

### 3.7 Computational cost comparison

The increase in speed of thermodynamic properties evaluation compared to cubic EOS is one of the primary motivations of this thesis, here we show an example of the speedup of computations. When  $(\rho, e)$  pair is evaluated from uniformly generated  $(P, T)$  combination in the nearcritical regime. For each pair iteration, 100000 loops/calls are executed and the slowest run time is tabulated. A contour plot of such calls is presented in figure 3.18. In comparison, the tabular lookup calls are 10 times faster than the calls to cubic EOS which in this case is chosen as Peng-Robinson EOS, it involves an iterative root-finding process and for this comparison its convergence tolerance is set to  $1E-06$ . Whereas in the tabular EOS, the searching algorithm (for a binary search tree it is  $\mathcal{O}(n)$ ) and interpolation scheme (here it is a bilinear) influence the lookup time, the adaptive tabulation is comparably denser in the near critical regime, because of high non linearity, comparatively higher lookup search times are observed for those regions as shown in the figure 3.18a



(a) Block based tabulated [AMR](#) lookup computational cost in  $\mu$  sec



(b) [PR EOS](#) evaluation computational Cost in  $\mu$  sec

Figure 3.18: Computational cost ( $\mu$  sec) comparison over Density ( $\frac{Kg}{m^3}$ ) evaluation calls.

### 3.8 Harmonic Acoustic wave test case

To test the computational performance and significance of using a tabular EOS, a one-dimensional acoustic wave propagation in supercritical fluid is considered, Using periodic boundary conditions at both sides a harmonic wave is initialized in supercritical conditions. The computational domain is  $x \in [0, 10]$  m and 100 grid points are used, giving a uniform grid spacing  $\Delta x = 0.1m$ . With fourth order accurate schemes in both spatial and temporal scales, both cubic (PREOS) and tabular EOS are used. The sound pressure level  $L_p = 23$  (dB) is calculated according to:

$$L_p = 20 \log_{10} \left( \frac{\Delta p}{p_{ref} \sqrt{2}} \right) \quad (3.8)$$

Where  $\Delta p$  is the amplitude of the pressure harmonic wave (Pa).

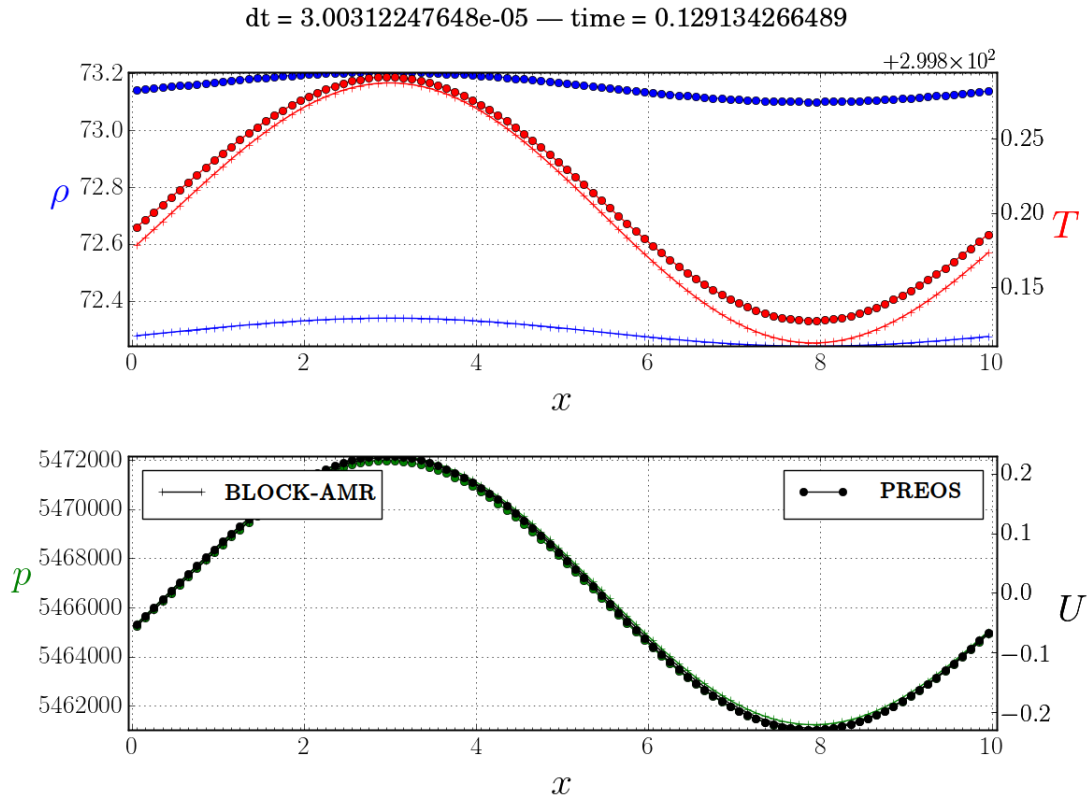


Figure 3.19: Harmonic Acoustic wave propagation; PREOS ( $\bullet$ ) vs AMR ( $-$ )



As shown in the figure 3.19, there is a distinction in the form of density by using accurate tabular real fluid EOS (Block-AMR) rather than conventional cubic PREOS. It is within the usually observed relative error, but it is important to mention that slight variations in temperature and pressure can cause significant change in density in the nearcritical regime.

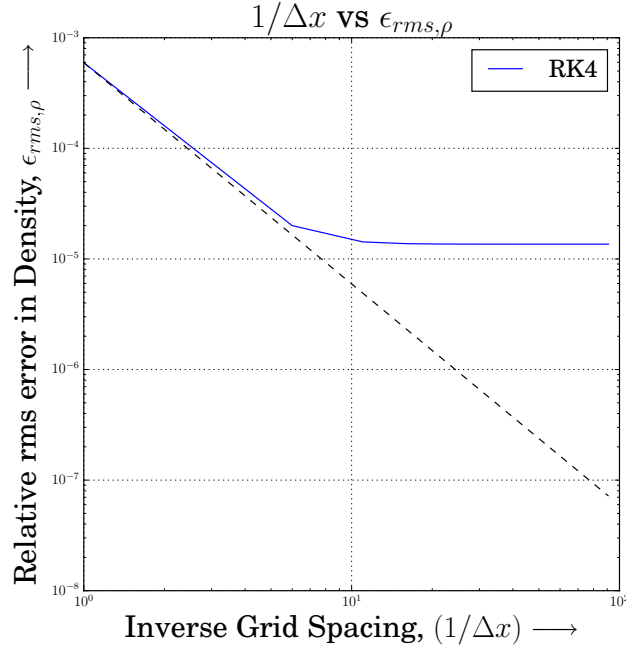


Figure 3.20: Grid convergence plot for the acoustic wave propagation case.

### 3.8.1 Computational cost comparison

To investigate how the choice of EOS affects the CFD computational cost (wall time in seconds) an experiment is performed with the above mentioned acoustic test case, the simulation is repeated using the available different EOS such as cubic (PREOS), tabular (Block-AMR, Bezier patch) and perfect gas law. The computational cost in seconds is plotted against increasing grid size in figure 3.22. It is important to mention here that the simulation code was written in Python, parallelized and ran on a CPU (Intel Core i5-4590 CPU 3.30GHz  $\times$  4) and utilized fourth order accurate schemes in both spatial and temporal scales. Hence the computational cost time is different from what is observed in a typical 1D simulation written in compiled languages such as C/C++. When we compare tabular

lookup based thermodynamic property evaluation and cubic polynomial equations. There are two important aspects that play a key role in influencing the time complexity in **AMR** and cubic **EOS**, they are 'lookup approach' and 'iterative solution method' respectively. The lookup approach here refers to the procedure to search/query the table for a given physical conditions (thermodynamic variables) and the iterative solution method refers to the repetitive root finding method such as Newton-Raphson solver, Bi-section method that is employed to obtain the root of the polynomial expression.

$$X_{n+1} = X_n - \left[ \frac{f(X_n)}{f'(X_n)} \right] \tag{3.9}$$

In the latter case, the time complexity depends on how good the initial guess value ( $x_0$ ) is, i.e., how close the initial guess value is to the root value. Further, during each iteration there will be a reduction of guess value into half  $\frac{x_0}{2}$  over and over. So approximately, it takes  $\frac{1}{2} \log_2(x_0)$  steps to reach within the vicinity of the supposed solution value of  $\sqrt{x_0}$ . After which, there will be a convergence process to zero-in onto the exact solution value. Here, the number of iteration steps that are required would be depended on the error tolerance value (not on the "Initial guess"). Thereby, the time complexity scales in the  $\mathcal{O}(\log(n))$

In the case of lookup search/query, the conventional data structures are multidimensional array, binary tree, quad-tree. Only the multidimensional array has a least worst case time complexity in access ( $\mathcal{O}(1)$ ). Whereas for the binary, quad-trees, K-D tree the worst case time complexity in accessing an element is  $\mathcal{O}(n)$ . k-d tree has order of complexity  $\mathcal{O}(k.N^{1-1/k})$  in the worst case, where 'k' being the number of dimensions.

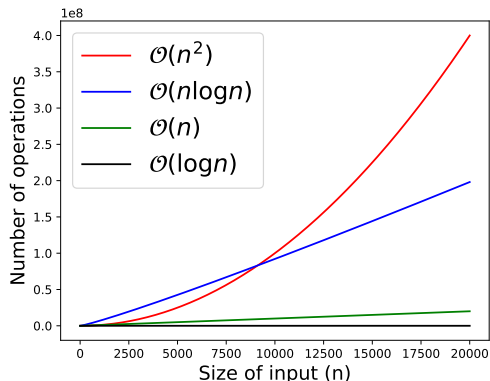


Figure 3.21: Big-O complexity plot

In our experiment we ran harmonic acoustic wave simulations with different types of EOS available namely, cubic, tabular and perfect gas. With a timestep size of  $\Delta t = 3.003E - 05s$ , we ran until time  $t = 0.129s$  starting with a grid size of 100 grid points, we increased grid size and plotted computational cost vs grid size as shown in figure 3.22. The key findings are that the cubic EOS involving iterative process of root finding had its computation cost vary quadratic with respect to the grid size. Upon an approximate logarithmic scaling analysis. the computation cost varies as

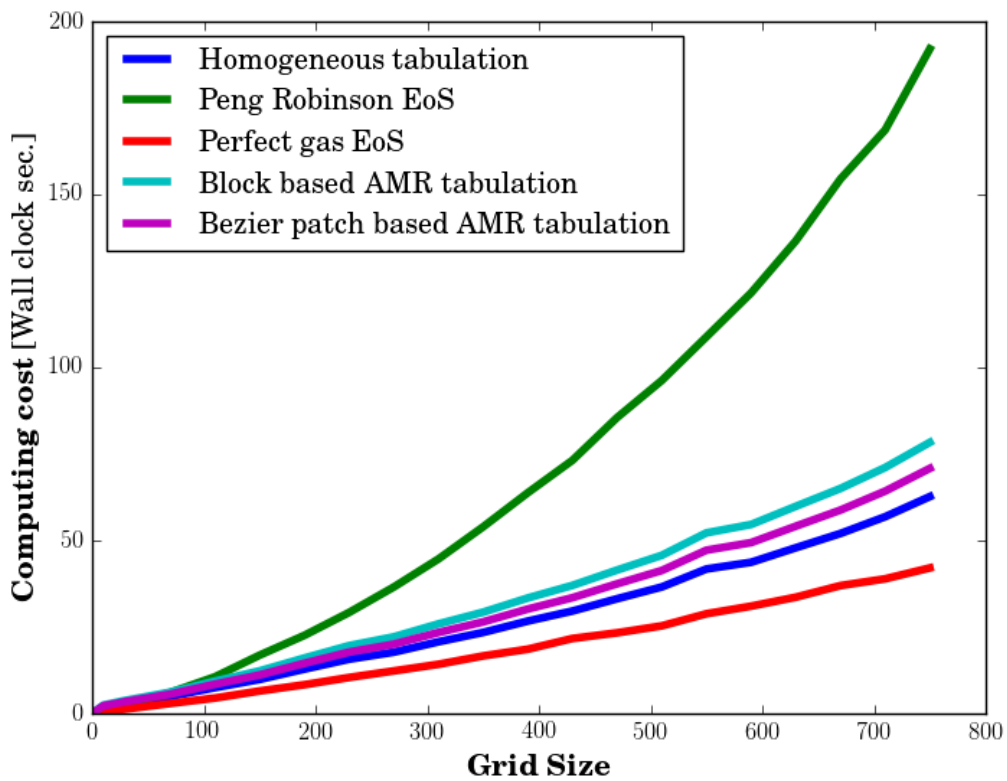


Figure 3.22: Computational cost ( $CC$ ) comparison with respect to grid size  $[Nx]$  for 1D-Acoustic wave problem with real fluid thermodynamics

$$\text{Computational cost}(CC) \approx \begin{cases} 0.000252[Nx]^2 + 0.058[Nx], \text{PREOS}, \\ 0.000023[Nx]^2 + 0.0394[Nx], \text{Block-AMR Tabulation}, \\ 0.000023[Nx]^2 + 0.0372[Nx], \text{Bezier-AMR Tabulation}, \\ 0.000023[Nx]^2 + 0.0352[Nx], \text{Homogeneous tabulation}, \\ 0.000003[Nx]^2 + 0.0324[Nx], \text{Perfect gas law} \end{cases}$$

### 3.9 Conclusions

Thermodynamic properties evaluation using tabulated EOS is implemented and various forms of tabulation such as homogeneous, adaptive grid based, higher degree polynomial based versions are developed, tested for their computational performance, accuracy and compared to the existing cubic EOS. In homogeneous tabulation the accuracy is heavily constrained to the properties range within which the table is being constructed, this is due to the limitations from cache memory perspective due to a uniform fine grid.

To avoid such limitations, adaptive tabulation methods are researched and efficient tabulation in the form of Block-structured AMR is presented. Although, it overcomes the table range limitations and being able to lookup with minimal computation cost, it requires additional storage in the form of predefined uniform index arrays (figure 3.5), overlapping index files. By avoiding the need to use any iterative solution methods to obtain thermodynamic properties, tabulation methods save a lot of computational cost as the experiment on the harmonic acoustic wave supports the claim.

# Chapter 4

## Implementation details of CFD Solver

In this chapter we provide a detailed overview of numerical methods involved in the TwoDThermoCode, the CFD solver extended as part of this thesis. The code is developed on top of Pyro2 [?], an Astrophysics simulation code. Additional features that was added for this thesis work are:

- Real fluid thermodynamic equation of state. (CoolProp database, Tabulated EOS)
- HLLC Riemann solver adapted to the real fluid thermodynamics
- Higher order time stepping scheme (RK4)

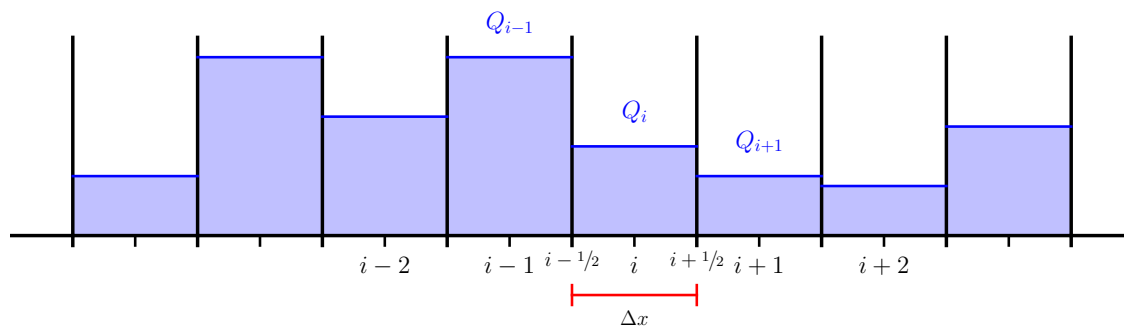


Figure 4.1: A representation of finite volume grid where an averaged value represents the variable throughout each cell [Figure reproduced based on [68]]

## 4.1 Conservative discretization of governing equations using Finite Volume methods

Assuming a conservation law in the form,

$$q_t + f(q)_x = 0 \quad (4.1)$$

where,  $q$  is a flux quantity,  $f$  is a flux function. We can discretize this in a one dimensional domain with the cell edges at  $[x_{i-1/2}, x_{i+1/2}]$ , as shown in the figure 4.1. The numerical solution ( $Q_i^n$ ) will be the volume average of the true solution  $q(x, t_n)$  over the cell. In simple terms, when we run a finite volume CFD solver we are not getting a pointwise value, instead we obtain the cell averaged value to the solution. By defining cell averages over the interval  $C_i = [x_{i-1/2}, x_{i+1/2}]$ , we can write the numerical solution as,

$$Q_i^n = \frac{1}{\Delta x} \int_{C_i} q(x, t_n) dx \quad (4.2)$$

The time rate of change of the true solution  $q(x, t_n)$  cell average is,

$$\frac{d}{dt} \int_{C_i} q(x, t) dx = - \int_{C_i} \frac{d}{dx} f(q(x, t)) dx = \underbrace{f(q(x_{i-1/2}, t))}_{\text{Flux in}} - \underbrace{f(q(x_{i+1/2}, t))}_{\text{Flux out}} \quad (4.3)$$

On integrating equation 4.3 in time, we obtain the time average value at time  $t_{n+1}$  as

$$\int_{C_i} q(x, t_{n+1}) dx = \int_{C_i} q(x, t_n) dx + \int_{t_n}^{t_{n+1}} [f(q(x_{i-1/2}, t)) - f(q(x_{i+1/2}, t))] dt \quad (4.4)$$

Using numerical fluxes, we can write the update as,

$$Q_i^{n+1} = Q_i^n - \frac{\Delta t}{\Delta x} [F_{i+1/2}^n - F_{i-1/2}^n] \quad (4.5)$$

$$\frac{Q_i^{n+1} - Q_i^n}{\Delta t} + \frac{F_{i+1/2}^n - F_{i-1/2}^n}{\Delta x} = 0 \quad (4.6)$$

Which resembles the conservation law equation 4.1 and we want to approximate the numerical fluxes ( $F_{i+1/2}^n, F_{i-1/2}^n$ ) for a given flux function  $f(q(x, t))$  and explicit time stepping scheme at cell interfaces  $[x_{i-1/2}, x_{i+1/2}]$ , in the form of,

$$F_{i-1/2}^n = \mathbf{F}(Q_i^n, Q_{i-1}^n), F_{i+1/2}^n = \mathbf{F}(Q_{i+1}^n, Q_i^n) \quad (4.7)$$

where  $Q_{i+1}^n, Q_i^n, Q_{i-1}^n$  are the cell averages. So, in simple terms we are trying to find the flux across the edges using the pointwise cell average values. This leads us to a classic Riemann problem of solving the hyperbolic conservation law with an initial constant piecewise data to obtain an analytic solution, explained in further sections (4.2, 4.3)

## 4.2 The Riemann problem

As we know that the Euler equations are a non-linear system of hyperbolic partial differential equations representing the conservation laws, obtaining an exact solution is a complex task, yet Godunov [23] proposed a scheme to obtain exact solutions to such governing equations using the Riemann problem, which can be defined as an Initial Value Problem for the hyperbolic conservation laws 4.1

$$U_t + (F(U))_x = 0 \quad (4.8)$$

where,

$$U = \begin{pmatrix} \rho \\ \rho u \\ \rho E \end{pmatrix} \quad F(U) = \begin{pmatrix} \rho u \\ \rho u^2 + p \\ \rho u E + up \end{pmatrix} \quad (4.9)$$

with initial conditions,

$$U(x, t = 0) = \begin{cases} U_l & x < 0 \\ U_r & x > 0 \end{cases} \quad (4.10)$$

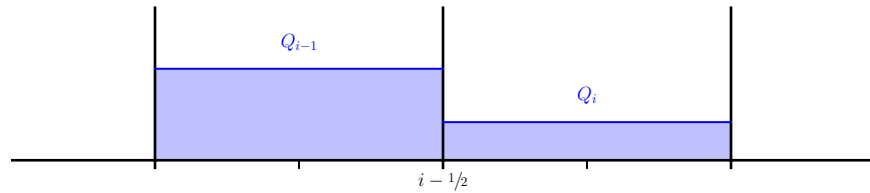


Figure 4.2: At time  $t = 0$ . (set from initial condition)

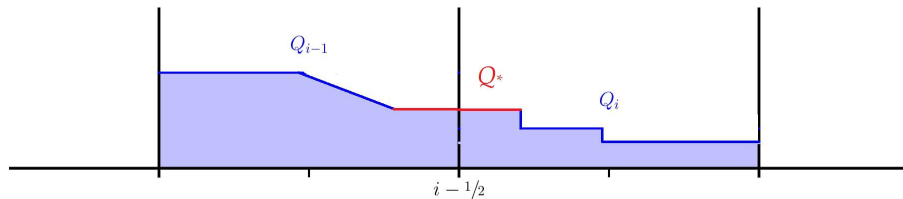


Figure 4.3: At time  $t > 0$

After time  $t > 0$ , if we look at the solution along the red line highlighted in figure, due to the initial condition assumed, the solution  $Q^*$  will be constant for a small finite amount of time  $\Delta t$ . We use this  $Q^*$  to evaluate the numerical flux at the interface, i.e.,  $F_{i-1/2} = f(Q^*)$ . This process is known as the classical Godunov [23] approach to solve the hyperbolic conservation law. Although it may be able to resolve shocks and rarefactions, it does so by solving conservation law at each and every cell's interface causing huge computational overhead. So, a typical CFD code doesn't employ Godunov's approach to solve Riemann problem.

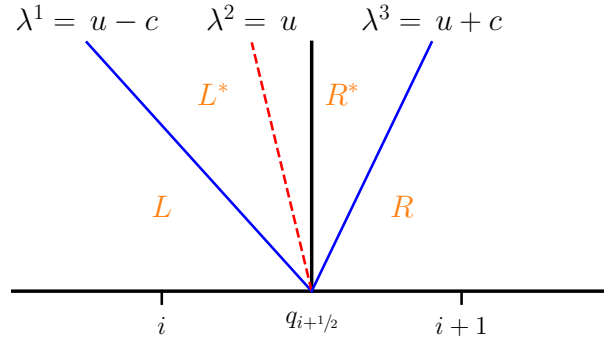


Figure 4.4: Representation of three wave patterns ( $\lambda^{(+)}$ ,  $\lambda^{(o)}$ ,  $\lambda^{(-)}$ ) of approximate Riemann solution, 4 different regions. Where x-axis represents space and the y-axis represents time. [Figure reproduced based on [68]]

The Riemann problem has three waves as shown in the figure 4.4, corresponding to the three eigenvalues of the matrix  $\lambda^1, \lambda^2, \lambda^3$  and the these waves separate four constant states namely  $L, L^*, R^*, R$ . The region between two non-linear waves ( $L, R$ ) is known as "Star Region" [68], which contains the contact discontinuity. If the flow is supersonic, the two non-linear acoustic waves can be either shocks or rarefactions leading to four possible combinations of wave patterns as shown in figure 4.6.



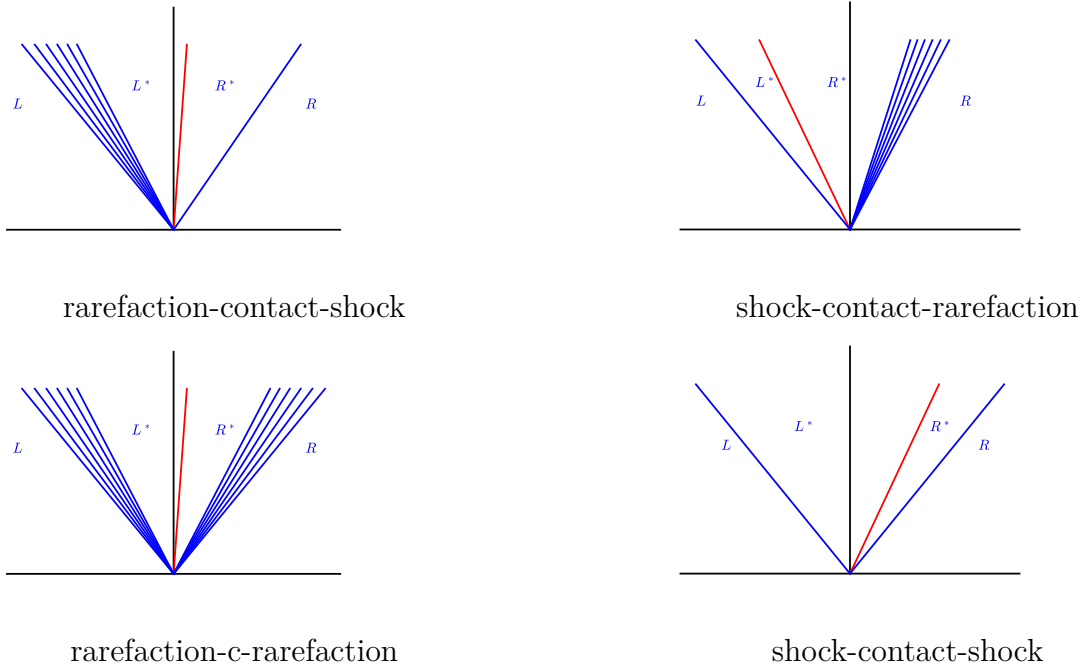


Figure 4.5: The possible wave pattern combinations to the solution of Riemann problem. (Figure reproduced from [68])

### 4.3 Riemann solvers with real fluid thermodynamics

At the heart of many finite volume compressible CFD codes is a Riemann solver, using which the numerical inviscid fluxes are computed. The Riemann problem described in 4.9 with suitable boundary conditions 4.10, can be solved by an explicit conservative method as in equation 4.3. In which the flux terms  $F_{i+1/2}^n$  is classified into two types based on the approach in which the wave propagation information is being used, either explicitly or implicitly. They are Godunov or Upwind type fluxes and non-upwind or centered fluxes. In the Godunov type fluxes we use the prior wave propagation information explicitly. Although we know how to obtain an exact solution to the Riemann problem using Godunov method, the reason why we introduce an approximate Riemann solver is because of the requirement to solve the conservation law at every cell interface over each iteration thereby making the solution computationally expensive. According to Toro[68], there are two ways to get an approximate solution to the Riemann problem, the first one involves obtaining an "approximation to the numerical flux" and the second is to obtain an approximate interface

state and solve for the flux function at that state. The methods HLL, HLLC solvers are categorized in this manner.

### 4.3.1 HLL Riemann solver

Proposed by Harten, Lax, Van Leer[29], this solution method involves a direct approximation of the flux at the interface state (single state in between two cells). The solution is assumed to be in a wave configuration consisting of two waves separating three constant states. The approximating numerical interface flux is given [68] as,

$$Q_{i+\frac{1}{2}} = \begin{cases} Q_L & 0 \leq S_L \\ Q_{hll} & S_L \leq 0 \leq S_R \\ Q_R & 0 \geq S_R \end{cases} \quad (4.11)$$

Here  $S_L, S_R$  are the wave speeds of two regions that separate star region denoted in figure 4.4.  $Q_L, Q_R$  are left, right fluxes and  $Q_{hll}$  is the HLL intercell flux calculated as,

$$Q_{hll} = \frac{S_R F_L - S_L F_R + S_L S_R (Q_R - Q_L)}{S_R - S_L} \quad (4.12)$$

### 4.3.2 HLLC Riemann solver

By restoring the absent contact and shear waves in the Euler equations, the HLLC scheme ('C' refers to Contact) is a modification of HLL scheme, and it is a three wave model consisting of  $S_L, S_*, S_R$ . The solution in the star region consists of two intermediate fluxes  $Q_{*L}, Q_{*R}$ . Thereby, the HLLC intercell flux is calculated [68] as,

$$Q_{i+\frac{1}{2}} = \begin{cases} Q_L & 0 \leq S_L \\ Q_{*L} & S_L \leq 0 \leq S_* \\ Q_{*R} & S_* \leq 0 \leq S_R \\ Q_R & 0 \geq S_R \end{cases} \quad (4.13)$$

The  $Q_{*L}, Q_{*R}$  fluxes are calculated as,

$$Q_{*L} = Q_L + S_L(U_{*L} - U_L) \quad (4.14)$$

$$Q_{*R} = Q_R + S_R(U_{*R} - U_R) \quad (4.15)$$

Here, the  $U_{*L}, U_{*R}$  denotes the two separate regions of star state. From equation 4.15

$$U_{*\omega} = \begin{pmatrix} \rho_{*\omega} \\ \rho_{*\omega} u_{*\omega} \\ \rho_{*\omega} E_{*\omega} \end{pmatrix} = \rho_{*\omega} \begin{pmatrix} S_{\omega} - u_{\omega} \\ S_{\omega} - S_* \end{pmatrix} \begin{pmatrix} 1 \\ S_* \\ (\frac{E_{\omega}}{\rho_{\omega}} + (S_* - u_{\omega})[S_* + \frac{\rho_{\omega}}{\rho_{\omega}(S_{\omega} - u_{\omega})}]) \end{pmatrix} \quad (4.16)$$

Here,  $\omega \in L, R$  and for a given star region pressure, velocity  $u_*$ ,  $p_*$ , the wave speeds ( $S_L, S_R, S_*$ ) are computed as follows,

$$S_L = \begin{cases} u_L - c_L & p_L \geq p_* \\ u_L + \frac{p_L - p_*}{\rho_L(u_L - u_*)} & p_L \leq p_* \end{cases} \quad (4.17)$$

$$S_* = u_* \quad (4.18)$$

$$S_R = \begin{cases} u_L + c_L & p_R \geq p_* \\ u_R + \frac{p_R - p_*}{\rho_L(u_R - u_*)} & p_R \leq p_* \end{cases} \quad (4.19)$$

### 4.3.3 Pressure oscillations in Supercritical flows

Supercritical fluids have non-linear variation of thermophysical properties near the critical point, the simulations associated with such sort of phase transition from the sub-critical to super-critical, the so called 'Transcritical' simulations always experience spurious pressure oscillations that inhibit the ease of modelling the fluid flow in these thermodynamic conditions. In the past decade there has been a significant improvement in tackling this problem especially from the numerical scheme perspective. Outlined below are some of those approaches.

*Double flux method:* The double flux method was first proposed by Abgrall[3] for a simple numerical scheme with perfect gas assumption, later this was extended into the real fluid simulations[43],[44]. In the context of these real fluid simulations, the spurious pressure oscillations appear when the specific heat ratio ( $\gamma$ ) and internal energy is not constant over the cell, therefore by keeping this specific heat constant in both space and time during the time integration over each time step and each individual cell we can restrain these oscillations. Also, in the primitive variables have to be updated in each sub-step of the time integration.

*Additional transport equations:* When simulating reacting compressible flows or multi-fluid flows, the pressure oscillations are frequent at the contact interfaces, even though

with the help of high order accurate Riemann solvers we can capture the shocks and be able to resolve the discontinuities in the case of single phase fluid. Johnsen and Colonius[34] presented a novel way of oscillation suppressing interface-capture methods based on Roe solver reconstruction of the primitive variables  $(\rho, u, p)$  so as to preserve the continuity and thereby eliminate the pressure imbalance in each cell. They extended these methods to the higher-order accurate Weighted Essentially Non-Oscillatory (WENO) schemes and HLLC approximate Riemann solver[34]. They also included a stiffened equation of state to model the individual components of the multiphase fluid. Shyue[57] used a quasi-conservative formulation of specific-heat ration ( $\gamma$ ) model (conservative, primitive equations that are rewritten in terms of the specific heat) and volume-fraction models (fraction of each liquid, gaseous component in each cell) to model multiphase fluids with a high resolution wave propagation method.

*Baer-Nunziato type models:* Contrary to the belief that having a strong non-linear variation in thermodynamic properties would lead to pressure oscillations Pantano [47] claims that the convexity of the equation of state is not the source of oscillations but rather the discontinuities that present across various interfaces cause these oscillations and no matter how non-linear the density due to the equation of state, these pressure oscillations are not easy to eliminate not even under the grid refinement. Thus causing a "violation of mechanical equilibrium". In their work, using a conservative formulation of the compressible Euler equations, an extra evolution equation[47] is added to the governing equations in order to carry the nonlinearity of the equation of state. *Entropy stable hybrid scheme:* Aiming at the transcritical turbulent flows, this method[44] uses double flux method[3, 44, 43] for the flux reconstruction and adaptively couples the higher-order non-dissipative and lower-order dissipative finite volume schemes with the help of a relative solution sensor, that flags the cells, usually that are characterized by large density gradients. *Non-conservative pressure formulation:* In order to maintain the pressure equilibrium across the fluid interfaces in the case of mixing, Terashima and Koshi[63, 64] proposed a pressure evolution based governing equation that will reconstruct the pressure and velocity across the interfaces instead as in the fully conservative method of evolving the total energy. Their approach[63, 64] also employed consistent numerical diffusion terms as their scheme was based on finite-difference formulation. Matheis and Hickel[45] addressed this oscillations problem in transcritical and supercritical fluid mixing in the case of a high pressure liquid-fuel injection[45]. They replaced the quasi-conservative, total energy based governing equations to non conservative pressure evolution equation[64]. They compared the both formats i.e., quasi-conservative and fully conservative simulations including the effect of diffusion induced pressure variation[45] solving for energy and pressure evolution equations respectively.

*Ghost fluid methods:* This particular method[20] focus heavily towards the oscillations at the material interfaces, by dividing the computational domain into individual fluids the Ghost fluid method[20] is developed based on a Eulerian scheme that treats the material interfaces in a Lagrangian format they used a level set function[20] to track the material interface along with the ghost cells keeping a constant pressure and velocity across the fluid interfaces as they remain constant irrespective of type of fluid. Thus, the scheme[20] is non-conservative at the material interface.

#### 4.3.4 Freezing the specific heat capacity ratio ( $\gamma$ )

The original HLLC approximate Riemann solvers is formulated based on a constant specific heat ratio ( $\gamma$ ), as we know that the thermodynamic properties vary rapidly as we approach the near critical temperature, pressure making the perfect gas assumption invalid. Also, there is a fundamental assumption[68] that the wave speed estimates that are based on speed of sound are valid for a given equation of state. Hence, the equation of state would only be needed to compute the speed of sound thereby the wave speeds (both left & right). In order to incorporate the real fluid effects into the HLLC solver, there are two solutions, one[4] is carrying thermodynamic index  $\gamma_e$  using an evolution equation and another is to carry  $\rho e$  which is calculated at the interface states in addition to the primitive variables  $\rho, u, p$ . Although this approach overfits the thermodynamics, it prevents the need to calculate  $\gamma_e$  as described in the first method. In the current project, the Colella and Glaz method[18] is used and the effective thermodynamic index is written as,

$$\gamma_e = \frac{p}{\rho e} + 1 \quad (4.20)$$

And then the characteristics for the Euler equations are solved where the speed of sound is calculate using the above mentioned thermodynamic index in perfect gas assumption for a given cell. A 1D advection simulation is setup to validate the significance of this alternative numerical technique, Note that the Euler system is solved for all test cases in this subsection, which enables a direct comparison with exact analytical solutions. The test case involves nitrogen as the working fluid. The pressure is set to 5 MPa, which is above the critical pressure of nitrogen. The computational domain is  $x \in [0, 1]$  m and a uniform mesh with 512 grid points is used. Periodic boundary conditions are applied. The initial conditions involve a sharp jump of density, given as

$$\rho = \begin{cases} 800 & 0.25 \leq x < 0.75 \\ 144.3, & otherwise \end{cases} \quad (4.21)$$

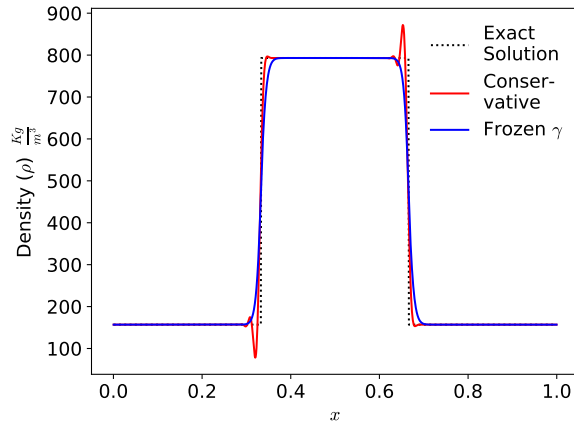


Figure 4.6: Density plot for the 1D real fluid Nitrogen advection case with initial condition as a sharp jump. Mesh size = 512.

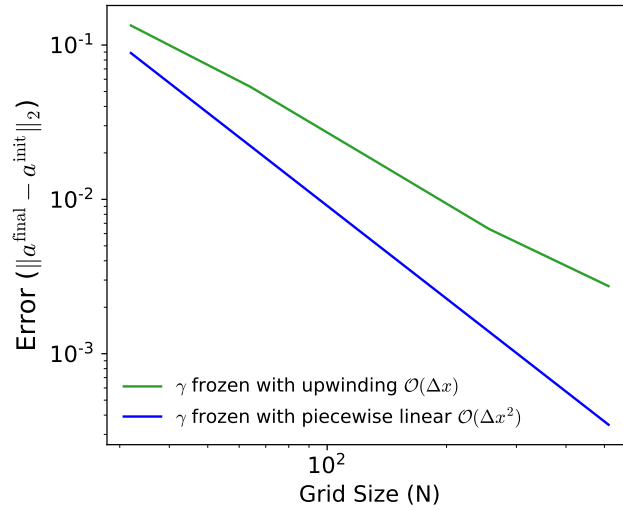


Figure 4.7: Convergence plot for the 1D real fluid Nitrogen Advection case with initial condition as a sharp jump. Mesh size = 512.

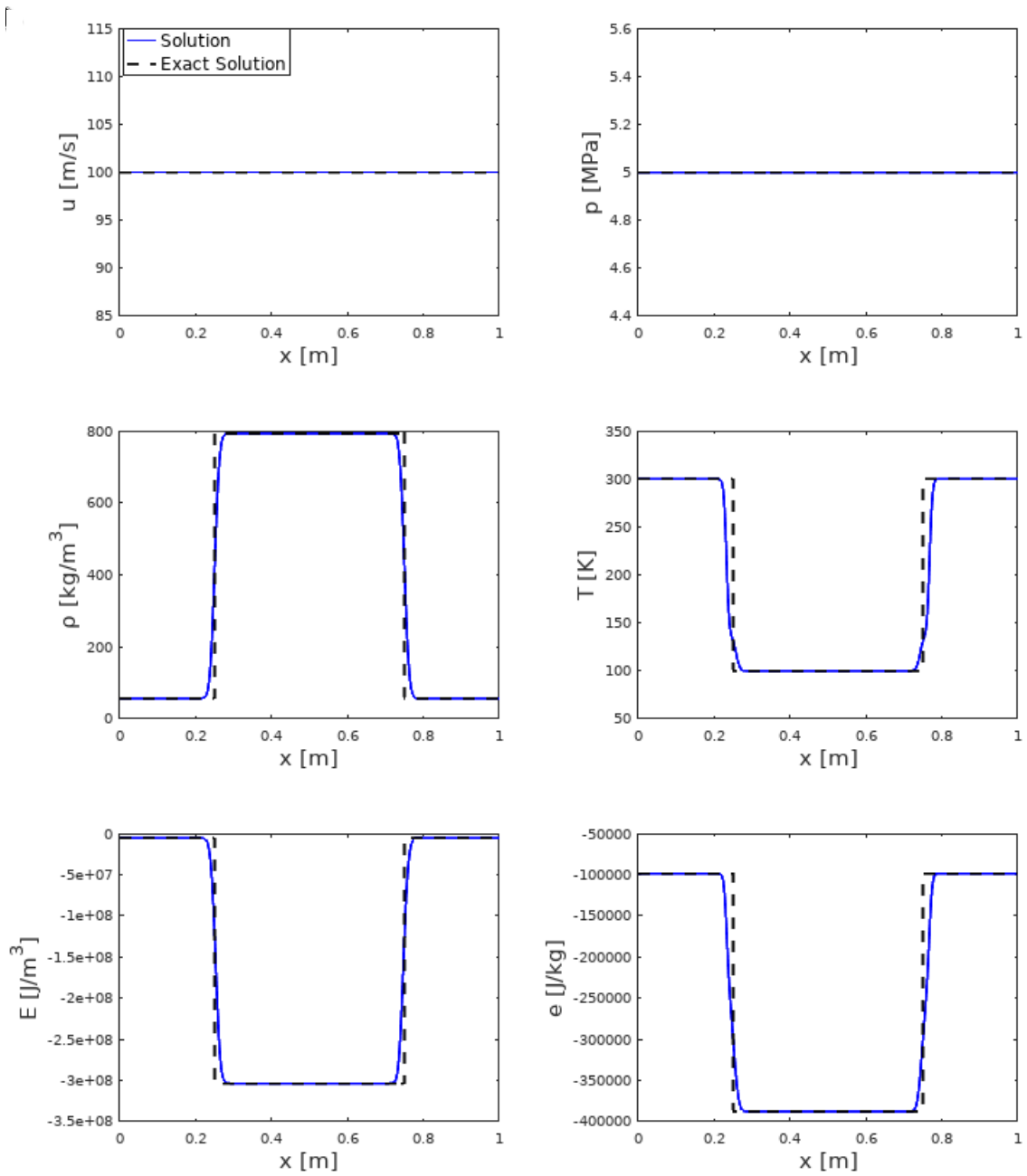


Figure 4.8: Plots of Density, velocity, temperature, total, internal energy for the 1D real fluid Nitrogen advection case with initial condition as a sharp jump. Mesh size = 512.

## 4.4 Fourth-order Runge-Kutta scheme (RK4)

The time advancement method is followed from [44], and implemented as described below.

- Compute the frozen  $\gamma$ , total energy across all the cells in mesh and store it to keep it as a constant over the entire timestep, across each cell.
- Starting with a defined conservative state, evaluate a single substep by reconstructing the fluxes and solving the Riemann problem with HLLC solver (adapted to real fluid thermodynamics by carrying the pre-computed gamma, total energy in above step).
- The RK substages are evaluated  $(k_1, k_2, k_3, k_4)$  during which the conservative variables are updated and after all stages of time integration are completed, a final update where all the substages are summed i.e.,  $(k_1 + 2k_2 + 2k_3 + k_4/6)$  and  $y_{t+1}$  is computed, primitive variables are updated from the conservative variables.

Here's the code snippet, that implements the above mentioned steps.

```
1     sos = myd.get_var("soundspeed")
2     rho = myd.get_var("density")
3     pres = myd.get_var("pressure")
4     gammaf = (sos**2)*rho/pres
5     e = eos.rhoe(rho, pres)/rho
6     e = e - (pres/(rho*(gammaf - 1.0)))
7     for s in range(4):
8         ytmp = rk.get_stage_start(s)
9         ytmp.fill_BC_all()
10        k = self.substep(ytmp, gammaf, e0S)
11        self.cons_to_prim()
12        rk.store_increment(s, k)
13    rk.compute_final_update()
```

Listing 4.1: RK4 with frozen specific heat ratio



## 4.5 Integration with tabulated thermodynamic equation of state

As we discussed in section 2.1 the governing equations are closed by an EOS (a function that maps the thermodynamic variables  $(rho, e) \rightarrow (P, T)$  or vice versa). Here, in the current CFD solver, EOS is used in context of transforming conservative variables  $(\rho, e)$  that are obtained by solving the flux vectors of conserved quantities  $(\rho, \rho\vec{u}, \rho E)$  to primitive variables  $(P, T)$ , which are necessary to calculate the flux. The code is implemented with both CoolProp, tabular EOS, for the latter the table generation and lookup process is described in chapter 2. It covers the super-critical, near-critical regime except the thermally equilibrium wet-region. In this thesis, for the transformation of conservative  $(\rho, e) \rightarrow$  primitive  $(P, T)$  and vice versa, we proceed with following method.

- During the solution initialization, to initialize conserved variables  $(\rho, e)$  based on  $(P, T)$  we use a multi-variate Newton-Raphson method with initial guess values (E.g;  $T = 300$  K,  $P = 101325$  Pa) to iterate and obtain corresponding density, internal energy values. This method is robust and efficient. Since, we run this iterative method only during the initialization, there is not much of an effort in terms of computation. Although we can use another table to lookup, i.e.,  $(\rho, e)$  tabulated in  $(P, T)$ , storing it in cache memory during runtime would make it ill-efficient.
- As we know that for a typical conservative finite volume method of Godunov type, once we set the conservative variables based on initial conditions at time,  $t = 0$ , we need not to transform  $(P, T) \rightarrow (\rho, e)$  at any point later on in the simulation, since the  $(\rho, e)$  pair is directly evolved from the numerical method thereby always known. So, we directly lookup in the pre-loaded EOS tabulated in  $(\rho, e)$

Although this method is robust and accurate, a relatively simpler approach is to use thermodynamic database (CoolProp NIST) functions directly. But, this can cause severe bottleneck when executed thousands of times during the higher order schemes. It is important to mention here that the previously mentioned inversion i.e.,  $(\rho, e) \rightarrow (P, T)$  using multi-variate Newton method is not valid in the wet-steam region due to the fact that the pressure and temperature become dependent variables and coupled by a relation.  $p = p_{saturation}(T)$ , here  $p_{saturation}(T)$  is saturation pressure at a given temperature T.

# Chapter 5

## Evaluation of tabulated supercritical thermodynamics

### 5.1 Real fluid thermodynamics significance in shock tube flow at supercritical conditions

Shock tube problems are of great use when it comes to validating numerical codes/schemes for various characteristics like shock capturing, dissipation, preventing oscillations etc. A fluid is said to be in a supercritical state, if its thermodynamic temperature and pressure are above its critical point. Since it is difficult to use experiments to validate the supercritical flow behaviour, often times simulations are widely used. Especially in the cases of supercritical fuel injection in diesel engines, mixing and injection of cryogenic propellants in rocket engines, or organic Rankine cycle turbines. The present chapter aims at understanding the influence of supercritical, near-critical, subcritical thermodynamic conditions on the shock tube problem and more specifically the shock wave, expansion fan locations in each case. It is well known that supercritical fluids have special thermodynamic properties by having a liquid like high density and gas like high diffusivity. In the proximity of critical point, specific heat increases drastically as shown in the figure 1.2 which causes a large expansion even for small increase in temperature. A perfect gas law cannot capture such non-linear variation thereby an accurate calculation of thermodynamic properties is required. There have been several attempts at studying shock tube flows in supercritical and nearcritical conditions. Near the critical point, the compression due to shock introduces a phase transition causing subsequent thermo-physical property changes resulting in an occurrence of non-classical waves [6]. The application of accurate equation of states to

the gas dynamics simulation revealed the formation of non-classical supersonic wave[27]. Arina [7] developed a numerical method for investigating supercritical fluid flow in nozzles, shock tube using cubic equation of states. Terashima[63],[64] presented a high resolution numerical method for simulating supercritical flows with large density variations and was able to resolve the flow interfaces over a wide range of scales. There have been several extensions of Riemann solvers to the real fluid thermodynamic conditions, specifically for van der Waals EOS [47],[6], stiff-gas EOS[4], cubic EOS[51].

### 5.1.1 Fundamental derivative of gas dynamics

In the process of understanding the key implications of using a real fluid thermodynamic equation of state for gas dynamics problems, we need to discuss about an important variable known as the "fundamental derivative of gas dynamics"[65, 46] which governs the non-linear dynamics of compressible fluids. It relates the change in speed of sound with reference to the change in density [65] at a given constant entropy, it is mathematically expressed as,

$$\tau(\rho, c) \equiv 1 + \frac{\rho}{c} \left( \frac{\partial c}{\partial \rho} \right)_s = \frac{\nu^3}{2c^2} \left( \frac{\partial^2 P}{\partial \nu^2} \right)_s \quad (5.1)$$

where,  $\rho$  is the density,  $c$  is the speed of sound,  $\nu = \frac{1}{\rho}$  is the specific volume and  $P$  is the pressure.

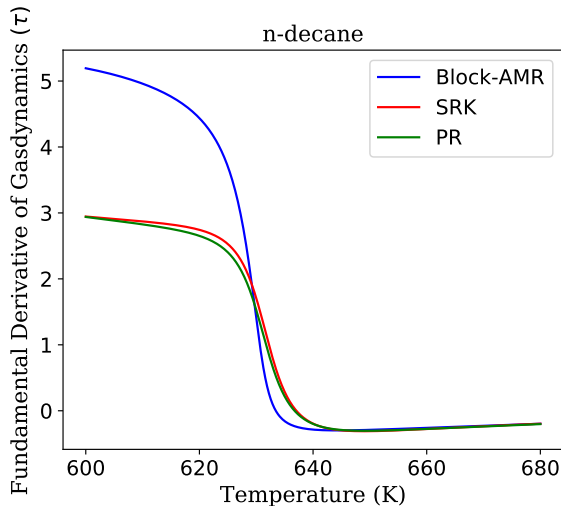


Figure 5.1: Fundamental derivative of gas dynamics evaluated using different equations of state for n-decane ( $C_{10}H_{22}$ )

This property denotes the direction in which the non-linearity propagates, when  $\tau > 0$  the non-linearity propagates in a conventional direction i.e., disturbances travel forward leading to compression shocks and when  $\tau < 0$ , the disturbances travel backward leading to rarefactions [65] and it is also been suggested [38] that this is required in order to satisfy the second law of thermodynamics. The fluids that fall into this category of negative non-linearity[46] are mainly comprised of dense gases, Bethe-Zel'dovich-Thompson fluids[38], complex organic compounds and the term "dense" here refers to the region close to its thermodynamic critical point or liquid-vapour curve.

| $\tau$         | Change in $\left(\frac{\partial c}{\partial \rho}\right)$            | Behaviour   | Remark            |
|----------------|--|---|-------------------|
| $\tau > 1$     | $\left(\frac{\partial c}{\partial \rho}\right) > 0$                  | Speed of sound increases with pressure  | Ideal             |
| $\tau = 1$     | $\left(\frac{\partial c}{\partial \rho}\right) = 0$                  | Speed of sound is constant and pressure varies linearly with density ( $\rho$ ) |                   |
| $0 < \tau < 1$ | $\frac{\rho}{c} < \left(\frac{\partial c}{\partial \rho}\right) < 0$ | Speed of sound decreases with pressure  | Real              |
| $\tau < 0$     | $\left(\frac{\partial c}{\partial \rho}\right) < \frac{\rho}{c}$     | Unusual   | non-classical[40] |

Table 5.1: Description of fundamental derivative of gas dynamics behaviour for different values [40, 65]

If we use a perfect gas EOS to compute this fundamental derivative ( $\tau$ ) where  $c = \sqrt{\gamma RT}$  we obtain,  $\tau = (\gamma + 1)/2$ , which is a constant for a given specific heat ratio ( $\gamma$ ). Which means, there is no chance for us to observe negative fundamental derivative ( $\tau < 0$ ) for any given fluid with perfect gas EOS assumption. This warrants the use of multiparametric EOS for such evaluations however as we discussed in chapter 2 with regard to the error associated with cubic EOS, using such equations to compute speed of sound and density would lead to further amplification of erroneous values. As can be seen in the figure 5.1 there is a significant difference in the fundamental derivative ( $\tau$ ) values computed using cubic EOS and tabular EOS

Before simulating such non-classical gas dynamics problems with real fluid EOS we try to obtain a validation for the CFD solver with the help of established testcases that employ a higher order accurate spatial and temporal schemes with real fluid EOS. The next two sections discuss the validation cases in two such test cases namely; Sod shock tube and Shu-Osher shock tube problems.

### 5.1.2 Sod Shock Tube Real gas validation case

Previously Terashima[64] carried out a 1D shock tube simulation under supercritical thermodynamic conditions, for this problem carbon dioxide is considered as the working fluid whose critical pressure, temperature and density [?] are  $T_{crit} = 304.22K$ ,  $p_{crit} = 7.37$  MPa,  $\rho_{crit} = 348.8 \frac{kg}{m^3}$ .

The initial conditions for this Sod shocktube simulation are

$$(\rho, u, p) = \begin{cases} (\rho_0, 0.0, 10p_0) & 0 \leq x \leq 0.5 \text{ m,} \\ (0.01\rho_0, 0.0, 0.1p_0) & 0.5 \leq x \leq 1.0 \text{ m} \end{cases}$$

Here,  $\rho_0 = \rho_{crit} = 348.8 \frac{Kg}{m^3}$ , and  $p_0 = p_{crit} = 7.37MPa$ , the grid consists of 2001 x 4 uniformly distributed cells.

The below plot is taken at  $t = 2.745E-04$  s. There is a good agreement in between the present solver and the reference simulation data obtained from the paper[63], no spurious pressure oscillations are found mainly due to the fact that the initial conditions are in supercritical regime. However, the transcritical case showed significant pressure oscillations.

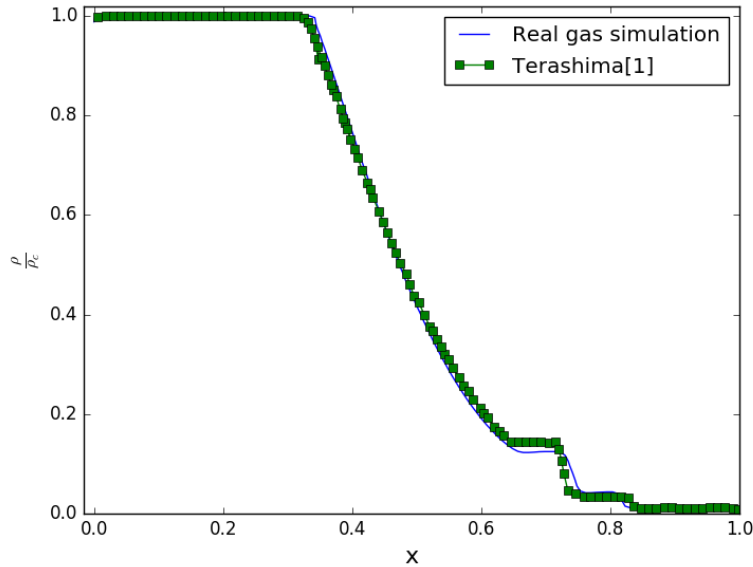


Figure 5.2: Sod Shock tube problem with real fluid thermodynamics Terashima validation case[63], Normalized Density  $\left(\frac{\rho}{\rho_{critical}}\right)$  vs distance plot taken at  $t = 2.745E - 04$  s, CFL = 0.4, Mesh size = 201, working fluid = Carbon dioxide.

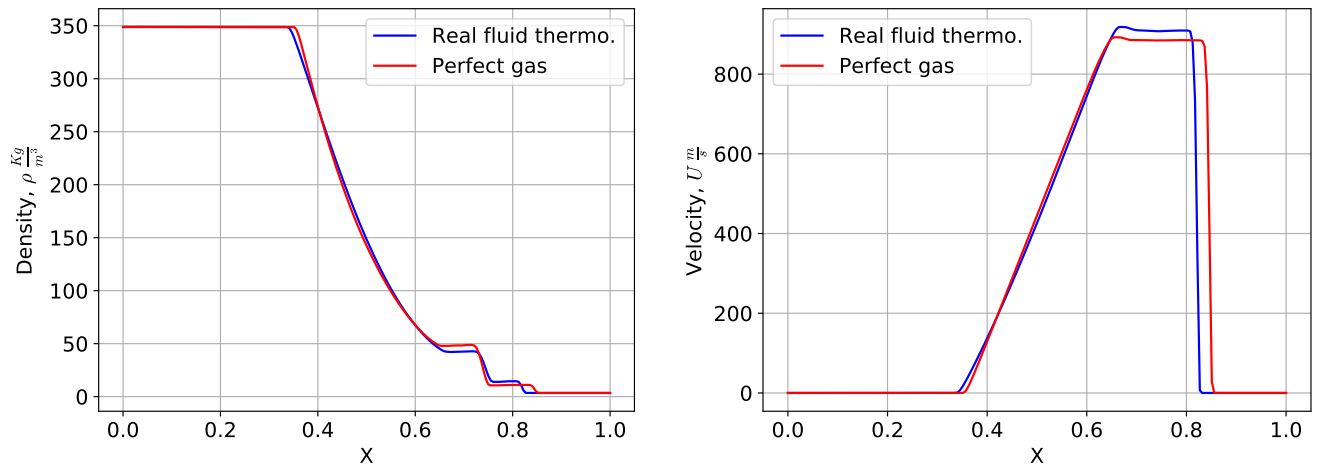


Figure 5.3: Sod Shock tube with real fluid thermodynamics compared with Ideal gas law

From the above figure on density distribution plot comparing real, ideal thermodynamics, we can see that the real gas case predicts the faster expansion fan when compared to the perfect gas case's expansion fan.

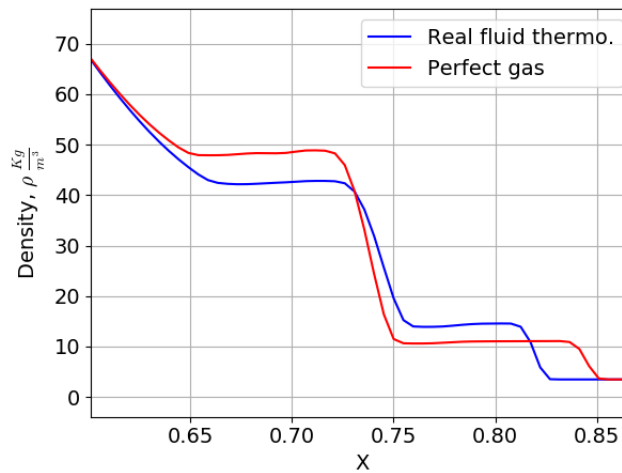


Figure 5.4: Sod Shock tube with real fluid thermodynamics compared with Ideal gas law

### 5.1.3 Shu-Osher shock tube real gas validation case

Another case used for reference is the Shu-Osher problem[64] that is modified to the supercritical regime (initial conditions). The purpose of testing with this case is to validate the code's dissipation characteristics. This test case is taken from Terashima[64], nitrogen is used as working fluid for this case and the initial conditions are,

$$(\rho, u, p) = \begin{cases} (3.857\rho_0, M_{ref}, 10.333p_0) & -0.5 \leq x \leq -0.4 \text{ m,} \\ ((1.0 + 0.2\sin 5x)\rho_0, 0.0, p_0) & -0.4 \leq x \leq 0.5 \text{ m} \end{cases}$$

Here,  $\rho_0 = 50.0 \frac{Kg}{m^3}$ ,  $p_0 = 4.0 MPa$  and  $M_{ref}$  is the reference sound speed on the left side. Similarly as previous case, the grid consists of 2001 uniformly distributed cells. The below plot is taken at  $5.903E-04$  s.

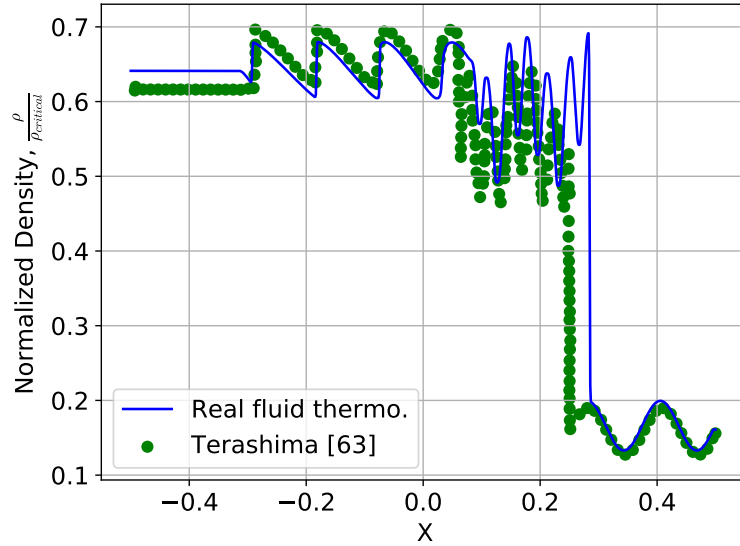


Figure 5.5: Shu-Osher shocktube problem with real fluid thermodynamics validation case[63], Normalized density ( $\frac{\rho}{\rho_{critical}}$ ) vs distance (x) plot

Figure 5.5 shows the normalized density ( $\frac{\rho}{\rho_{crit}}$ ) plot for the current real fluid modified solver vs the reference case. It is important to mention here that the reference case was obtained by smoothening the initial interface with an adjustable free parameter  $c_\epsilon = 3.0$ . This could be a reason for slight mismatch in the density profiles.

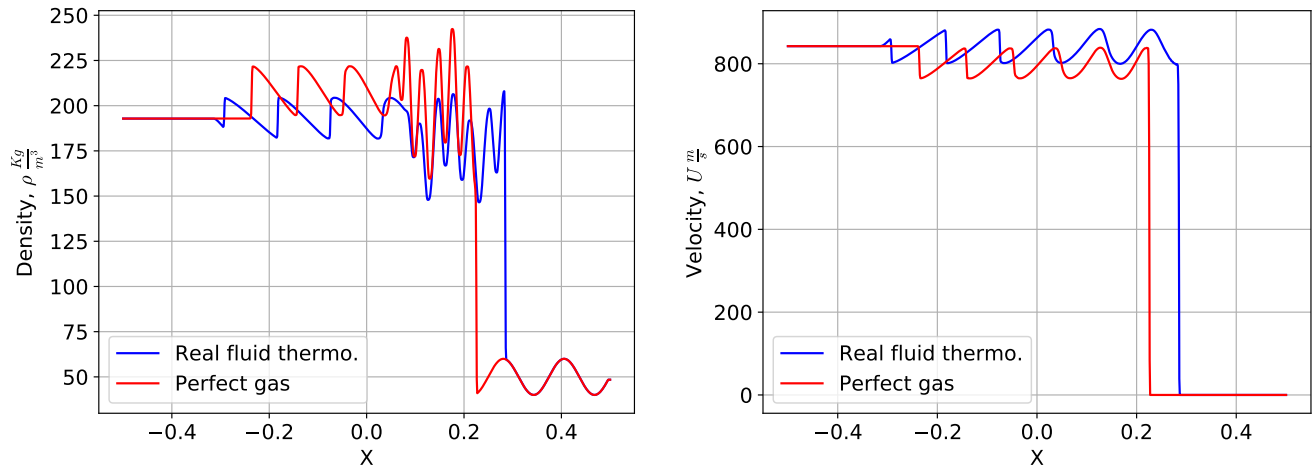


Figure 5.6: Shu-Osher shock tube problem comparison between real fluid thermodynamics and perfect gas assumption, density ( $\rho$ ), velocity plots vs distance. Mesh size ( $N$ ) = 201, working fluid is Nitrogen.

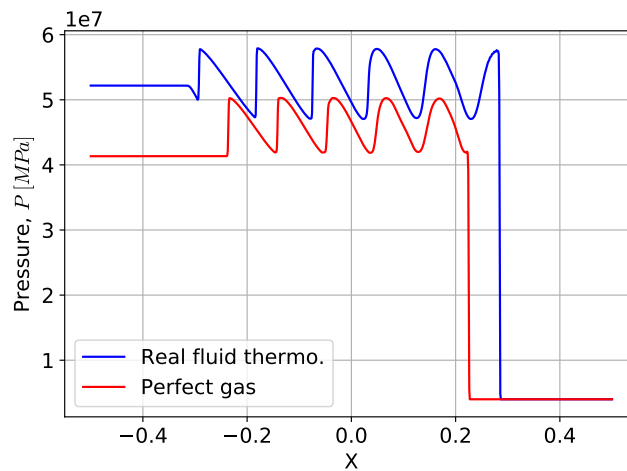


Figure 5.7: Shu-Osher shock tube problem comparison between real fluid thermodynamics and perfect gas assumption, Pressure ( $P$ ), vs distance plot. Mesh size ( $N$ ) = 201, working fluid is Nitrogen.



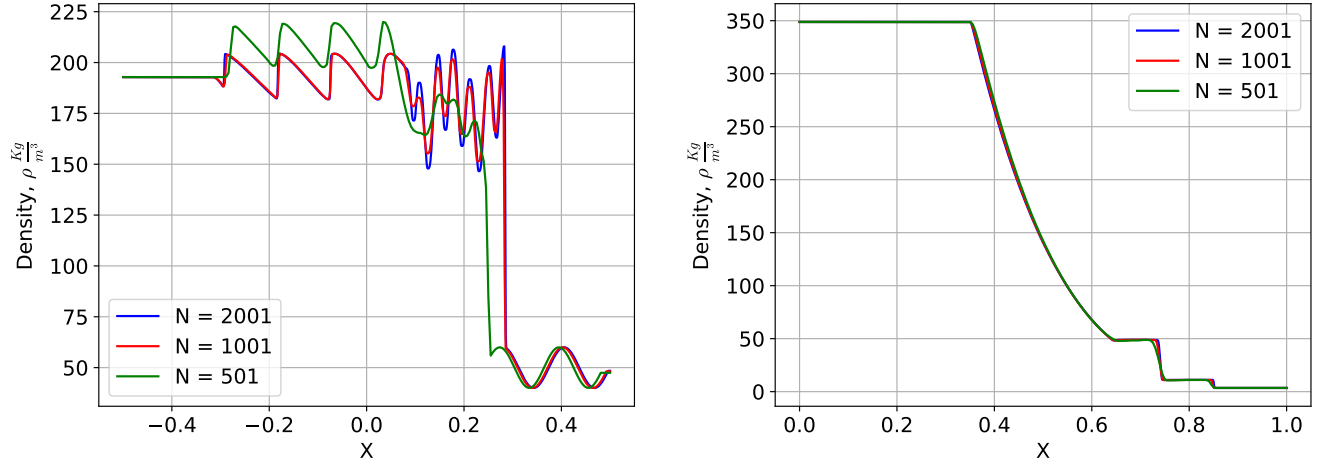


Figure 5.8: Mesh Independence density profile plot of Shu-Osher shock tube problem with real fluid thermodynamics, taken at  $t = 2.745E - 04$  s, CFL = 0.4 Mesh sizes (N) = 501, 1001, 2001,

### 5.1.4 Accurate real-fluid thermodynamics significance in shock tube problem

In order to study the real-fluid effects on shock tube wave propagation, three cases are simulated at supercritical, near-critical and the subcritical thermodynamic conditions. The initial conditions are shown in the table 5.2 for each case, in all the cases the left to right pressure ratios ( $\frac{p_{left}}{p_{right}}$ ) are set to 2 with an initial temperature ( $T$ ) of 310 K. Figure 5.10

Table 5.2: Initial conditions for Sod-problem.

|               | $P_{left}$  | $P_{right}$  | $P_{left}/P_{right}$ | T, K | $\rho_{left}$ | $\rho_{right}$ |
|---------------|-------------|--------------|----------------------|------|---------------|----------------|
| Supercritical | $4p_{cr}$   | $2p_{cr}$    | 2                    | 310  | 845           | 696            |
| Nearcritical  | $1.5p_{cr}$ | $0.75p_{cr}$ | 2                    | 310  | 610           | 134            |
| Subcritical   | $0.5p_{cr}$ | $0.25p_{cr}$ | 2                    | 310  | 76.9          | 34.4           |

shows the pressure, temperature, density distribution plots, in which the variables ( $\rho$ , P, T) are non-dimensionalized with respect to the initial values on left side. For the case of nearcritical, comparatively smaller pressure, temperature jump ( $\frac{p}{p_{left}}$ ) is observed at the

contact interface, while the density jump is largest. This can be attributed to the strong non-linear variation of thermodynamic properties in the proximity of critical point.

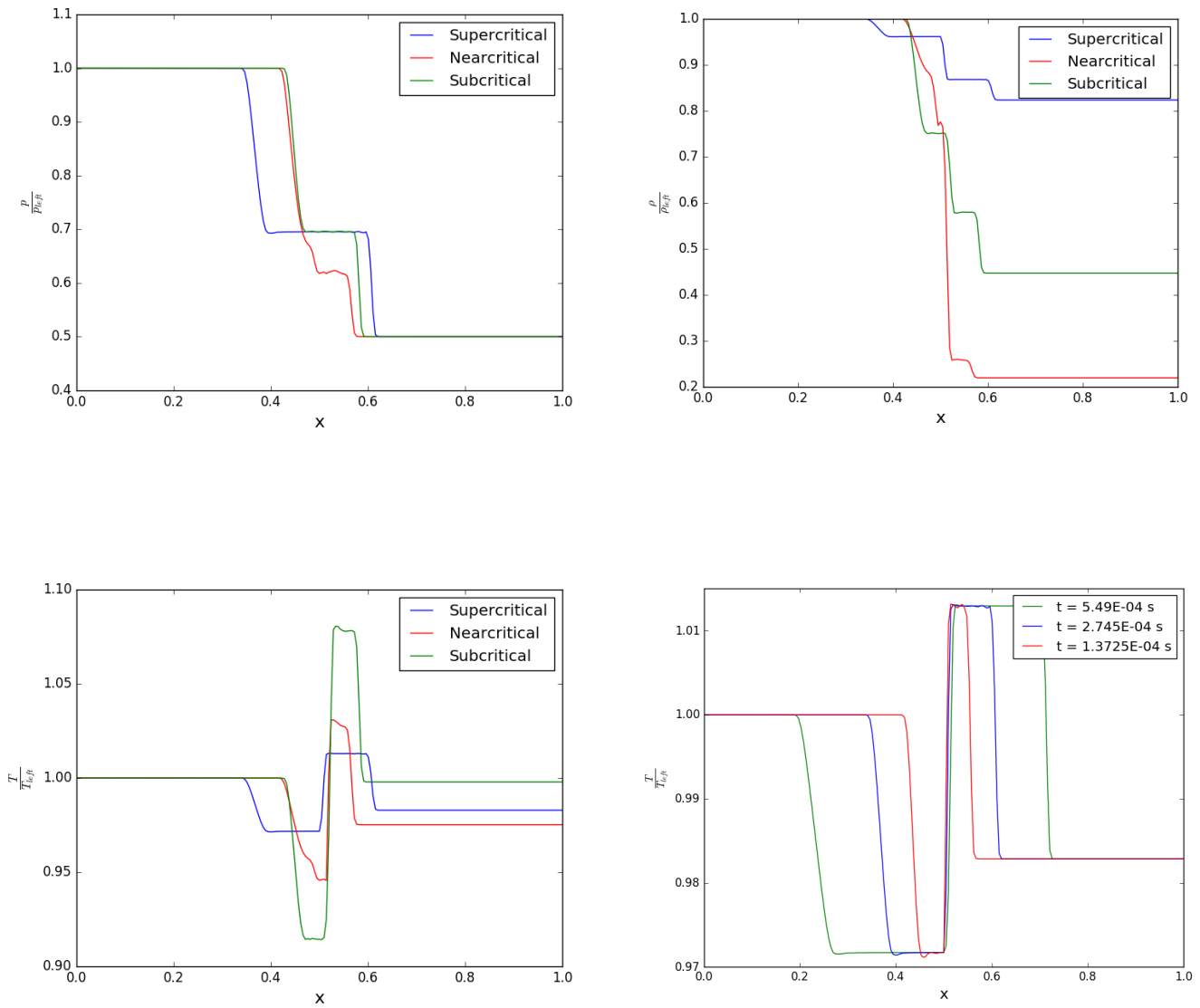


Figure 5.10: Plots of non-dimensionalized pressure, density, temperature and time histories of temperature profile for the near critical case . Here the variables are non-dimensionalized using the initial left side values.

A small temperature overshoot can be observed next to the contact discontinuity. We have discussed in section 4.3.4 on various numerical techniques to overcome this oscillations.

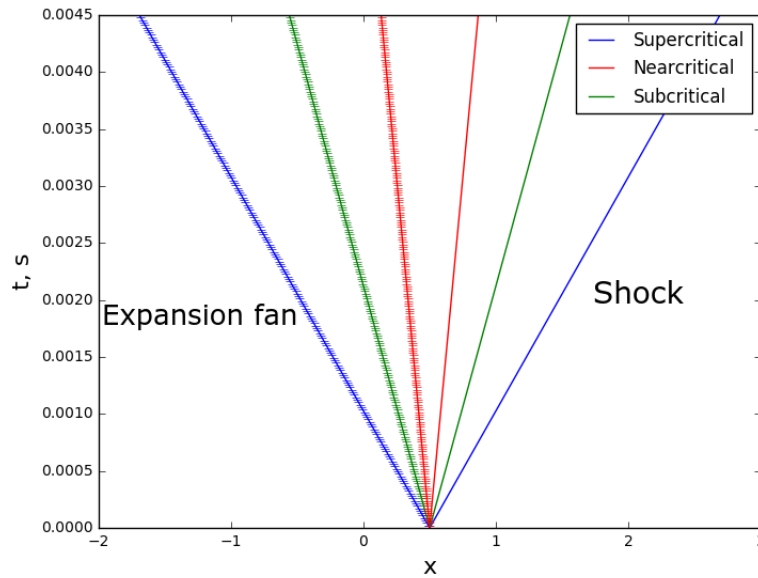


Figure 5.11: Sod shocktube problem with three different real fluid thermodynamics cases, plot of time histories of shock wave (—) , expansion fan (+++) locations.

Figure 5.11 showcases the shows time histories of the wave positions, in the real fluid supercritical case, both the shock and the expansion fan propagate faster than in the ideal gas assumption. Though, we couldn't verify whether this is purely due to the calculation mismatch in speed of sound or there is really a certain physical significance.

# Chapter 6

## Conclusions and Future Work

### 6.1 Conclusions

Paving way towards accurate real-fluid [CFD](#) simulations, this thesis provides improvised method in a key area, real fluid thermodynamic property evaluation using an adaptive tabulation, given the fact that the [CFD](#) simulations with complex thermodynamic systems involve expensive simulations due to the non-linear variation of thermodynamic properties and numerical scheme to model large density gradients. [CFD](#) solvers require [EOS](#) to transform the conservative variables to their primitive state. Typically in case of cubic [EOS](#) such as Peng-Robinson this is accomplished through an iterative algorithm to find the correct (P, T) primitive pair for given conservative pair ( $\rho$ , e). The emphasis in obtaining such accurate thermodynamic information is due to the strong non-linear thermodynamic property gradients observed in the vicinity of critical point i.e., any minute changes in temperature or pressure drastically changes the density. As an alternative to such iterative algorithm, this thesis presented a detailed framework on Tabulated real fluid thermodynamics, starting from the simplest form of tabulation i.e., homogeneous or logarithmic tabulation to various other forms of tabulation such as Block structured adaptive mesh refinement [AMR](#) and Cubic Bezier surface based adaptive mesh refinement [AMR](#) are described and implemented in detail. When compared to the homogeneous tabulation, adaptive tabulation gives efficient memory storage according for a given user defined error tolerance, generating a homogeneous [EOS](#) table with very low error tolerances generates large sized storage files thereby limits the physical and thermodynamic range within which the table is going to support. When the tabular [EOS](#) is compared with the cubic equations in terms of accuracy, evaluation (look-up) speed there was a significant improvement. To

observe the speedup, the tabular EOS is implemented into a 1D CFD code and the grid size is scaled incrementally. The cubic EOS based simulations increased the computational cost non-linearly with respect to the grid size. To further investigate the significance of using real fluid thermodynamics in a gas dynamics simulation, a CFD code with higher order spatial and temporal schemes is developed and its numerical schemes were discussed in chapter 4. In order to overcome the oscillations at the sharp interfaces, a method in which the specific heat ratio is kept constant over a time step across each cell is discussed and implemented. It helped in damping the oscillations and enabled to obtain stable simulations. Later the process through which the adaptive tabular EOS is integrated into this CFD code is described. The relevance of computing accurate thermodynamic properties in gasdynamics problems is described in chapter 5 coupling both the tabulated thermodynamic equation of state framework and the CFD solver together, two test cases are used for this study, sod-shock tube and the shu-osher shock tube. A validation result is obtained by simulating a near critical testcase taken from Terashima [64]. Three different thermodynamic initial conditions namely subcritical, supercritical and near critical regions are simulated. It was found that the accurate thermodynamics evaluation resulted in a faster expansion fan compared to the perfect gas assumption. In the test case where three different thermodynamic conditions were tested, among them the nearcritical case had a significantly smaller temperature, pressure jumps, whereas the density jump was the largest among all the three cases. Also, the expansion fan was faster while using real fluid thermodynamics than perfect gas assumption.

## 6.2 Future Work

There are two aspects to which the current thesis can be extended, in terms of tabular EOS in present work there is no consideration of the region below the vapour dome (wet-steam region). Also, the present tabulation is limited to single phase, pseudo-pure fluids. Most of the industrial applications involve the use of mixtures, hence this current two dimensional tabulation can be extended into three dimensions, where the third dimension can be mixture fraction.

The second aspect is the uncertainty quantification, as with every EOS there will be an uncertainty associated within the output values, in the case of tabular EOS apart from a parametric uncertainty derived from the underlying empirical EOS there will be an additional model uncertainty due to the interpolation, tabulation. Understanding and quantifying this tabulation uncertainty is key to have a reliable simulation. In terms of test cases, the tabular EOS can be tested with Large Eddy Simulation (LES) softwares,

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# APPENDICES

# Appendix A

## A.1 Code repositories

The adaptive tabulation based thermodynamic equation of state code is available in git repository <https://github.com/spmuppar/Adaptive-Tabulated-real-fluid-thermo> and the cfd code developed for this thesis is available at <https://github.com/spmuppar/TwoDThermoCode>

## A.2 HLLC Riemann solver adapted for Real fluid thermodynamics module

```
1
2 SUBROUTINE riemann_HLLC_real(idir, qx, qy, ng, &
3                               nvar, idens, ixmom, iymom, iener, &
4                               lower_solid, upper_solid, &
5                               real_gamma, pres, U_l, U_r, F)
6
7 !implicit none
8 EXTERNAL real_gamma
9 EXTERNAL pres
10
11 integer, intent(in) :: idir
12 integer, intent(in) :: qx, qy, ng
13 integer, intent(in) :: nvar, idens, ixmom, iymom, iener
14 integer, intent(in) :: lower_solid, upper_solid
```

```

15
16 double precision, intent(inout) :: U_l(0:qx-1,0:qy-1,0:nvar-1)
17 double precision, intent(inout) :: U_r(0:qx-1,0:qy-1,0:nvar-1)
18 double precision, intent( out) :: F(0:qx-1,0:qy-1,0:nvar-1)
19 !F2PY (CALLBACK) real_gamma
20 !F2PY (CALLBACK) pres
21 !f2py depend(qx, qy, nvar) :: U_l, U_r
22 !f2py intent(in) :: U_l, U_r
23 !f2py intent(out) :: F
24
25
26 integer :: ilo, ihi, jlo, jhi
27 integer :: nx, ny
28 integer :: i, j
29
30 double precision, parameter :: smallc = 1.e-10
31 double precision, parameter :: smallrho = 1.e-10
32 double precision, parameter :: smallp = 1.e-10
33 double precision, dimension(0:1) :: temp
34 real :: temp1
35
36 double precision :: rho_l, un_l, ut_l, rhoe_l, p_l
37 double precision :: rho_r, un_r, ut_r, rhoe_r, p_r
38
39 double precision :: rho_star_l, rho_star_r, rho_star_l,
    rho_star_r
40 double precision :: u_star, p_star, c_star_l, c_star_r
41 double precision :: lambda_l, lambda_star_l, lambda_r,
    lambda_star_r
42 double precision :: W_l, W_r, c_l, c_r, sigma
43 double precision :: alpha
44 double precision :: eint_l, eint_r
45
46 double precision :: rho_state, un_state, ut_state, p_state,
    rhoe_state
47 double precision :: rho_avg
48 double precision :: Q, p_min, p_max, p_lr, p_guess
49 double precision :: factor, factor2

```

```

50 double precision :: g_l, g_r, A_l, B_l, A_r, B_r, z
51 double precision :: S_l, S_r, S_c
52 double precision :: c_avg
53
54 double precision :: U_state(0:nvar-1)
55 double precision :: HLLCfactor
56
57 nx = qx - 2*ng; ny = qy - 2*ng
58 ilo = ng; ihi = ng+nx-1; jlo = ng; jhi = ng+ny-1
59
60 do j = jlo-1, jhi+1
61   do i = ilo-1, ihi+1
62     ! primitive variable states
63     rho_l = U_l(i,j,idents)
64
65     ! un = normal velocity; ut = transverse velocity
66     if (idir == 1) then
67       un_l = U_l(i,j,ixmom)/rho_l
68       ut_l = U_l(i,j,iymom)/rho_l
69     else
70       un_l = U_l(i,j,iymom)/rho_l
71       ut_l = U_l(i,j,ixmom)/rho_l
72     endif
73
74     rhoe_l = U_l(i,j,iener) - 0.5*rho_l*(un_l**2 + ut_l**2)
75     eint_l = rhoe_l/rho_l !this will be negative
76
77     temp = [rho_l, eint_l]
78     temp1 = 0.0d0 + pres(temp)
79     p_l = temp1
80     !p_l = rhoe_l*(gamma - 1.0d0)
81     p_l = max(p_l, smallp)
82
83     rho_r = U_r(i,j,idents)
84
85     if (idir == 1) then
86       un_r = U_r(i,j,ixmom)/rho_r
87       ut_r = U_r(i,j,iymom)/rho_r

```



```

88     else
89         un_r      = U_r(i,j,iymom)/rho_r
90         ut_r      = U_r(i,j,ixmom)/rho_r
91     endif
92
93     rhoe_r = U_r(i,j,iener) - 0.5*rho_r*(un_r**2 + ut_r**2)
94     eint_r = rhoe_r/rho_r
95
96     temp = [rho_r, eint_r]
97     temp1 = 0.0d0 + pres(temp)
98     p_r = temp1
99     !p_r  = rhoe_r*(gamma - 1.0d0)
100    p_r = max(p_r, smallp)
101
102    ! define the Lagrangian sound speed
103    temp = [rho_l, p_l]
104    temp1 = 0.0d0 + real_gamma(temp)
105    gamma_l = temp1
106
107    temp = [rho_r, p_r]
108    temp1 = 0.0d0 + real_gamma(temp)
109    gamma_r = temp1
110
111    W_l = max(smallrho*smallc, sqrt(gamma_l*p_l*rho_l))
112    W_r = max(smallrho*smallc, sqrt(gamma_r*p_r*rho_r))
113
114    ! and the regular sound speeds
115    c_l = max(smallc, sqrt(gamma_l*p_l/rho_l))
116    c_r = max(smallc, sqrt(gamma_r*p_r/rho_r))
117
118    ! define the star states
119    pstar = (W_l*p_r + W_r*p_l + W_l*W_r*(un_l - un_r))/(W_l +
120    W_r)
121    pstar = max(pstar, smallp)
122    ustar = (W_l*un_l + W_r*un_r + (p_l - p_r))/(W_l + W_r)
123
124    ! now compute the remaining state to the left and right
125    ! of the contact (in the star region)

```

```

125  rhostar_l = rho_l + (pstar - p_l)/c_l**2
126  rhostar_r = rho_r + (pstar - p_r)/c_r**2
127
128  ! step II: wave speed estimate
129  ! estimate the nonlinear wave speeds
130
131  ! use the simplest estimates of the wave speeds
132  S_l = min(un_l - sqrt(gamma_l*p_l/rho_l),
133          un_r - sqrt(gamma_r*p_r/rho_r))
134  S_r = max(un_l + sqrt(gamma_l*p_l/rho_l),
135          un_r + sqrt(gamma_r*p_r/rho_r))
136
137  ! Eqn 10.70 in Toro
138  S_c = (p_r - p_l + rho_l*un_l*(S_l - un_l) -
139        rho_r*un_r*(S_r - un_r))/ &
140        (rho_l*(S_l - un_l) - rho_r*(S_r - un_r))
141
142  ! step III: Compute the HLLC flux
143  ! figure out which region we are in and compute the state
and
144  ! the interface fluxes using the HLLC Riemann solver
145  if (S_r <= 0.0d0) then
146    ! R region
147    U_state(:) = U_r(i,j,:)
148    call consFlux(idir, pres, idens, ixmom, iymom,
149                iener, nvar, U_state, F(i,j,:))
150  else if (S_r > 0.0d0 .and. S_c <= 0) then
151    ! R* region
152    HLLCfactor = rho_r*(S_r - un_r)/(S_r - S_c)
153    U_state(idens) = HLLCfactor
154    if (idir == 1) then
155      U_state(ixmom) = HLLCfactor*S_c
156      U_state(iymom) = HLLCfactor*ut_r
157    else
158      U_state(ixmom) = HLLCfactor*ut_r
159      U_state(iymom) = HLLCfactor*S_c
160    endif
161    U_state(iener) = HLLCfactor*(U_r(i,j,iener)/rho_r + &

```

```

162         (S_c - un_r)*(S_c + p_r/(rho_r*(S_r - un_r)))
163     ! find the flux on the right interface
164     call consFlux(idir, pres, idens, ixmom, iymom,
165                 iener, nvar, U_r(i,j,:), F(i,j,:))
166     ! correct the flux
167     F(i,j,:) = F(i,j,:) + S_r*(U_state(:) - U_r(i,j,:))
168     else if (S_c > 0.0d0 .and. S_l < 0.0) then
169         ! L* region
170         HLLCfactor = rho_l*(S_l - un_l)/(S_l - S_c)
171         U_state(idens) = HLLCfactor
172         if (idir == 1) then
173             U_state(ixmom) = HLLCfactor*S_c
174             U_state(iymom) = HLLCfactor*ut_l
175         else
176             U_state(ixmom) = HLLCfactor*ut_l
177             U_state(iymom) = HLLCfactor*S_c
178         endif
179         U_state(iener) = HLLCfactor*(U_l(i,j,iener)/rho_l + &
180             (S_c - un_l)*(S_c + p_l/(rho_l*(S_l - un_l))))
181
182     ! find the flux on the left interface
183     call consFlux(idir, pres, idens, ixmom, iymom, iener,
184     nvar, &
185                 U_l(i,j,:), F(i,j,:))
186     ! correct the flux
187     F(i,j,:) = F(i,j,:) + S_l*(U_state(:) - U_l(i,j,:))
188     else
189         ! L region
190         U_state(:) = U_l(i,j,:)
191
192         call consFlux(idir, pres, idens, ixmom, iymom, iener,
193     nvar, &
194                 U_state, F(i,j,:))
195     endif
196 enddo
197 enddo
198 end subroutine riemann_HLLC

```

### A.3 CFD code development git history for this study

